



TEMPERATURE DISTRIBUTION IN A COAXIAL RING TUBE  
DUCT FOR HTR APPLICATIONS

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# TEMPERATURE DISTRIBUTION IN A COAXIAL RING TUBE DUCT FOR HTR APPLICATIONS

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## ABSTRACT

A computer program has been developed to help optimize the design of a coaxial ring tubes type hot gas duct to be used in high temperature reactors (HTR). In this design the hot helium (950°C) flows in a large ceramic pipe and the cold helium returns to the reactor through the holes made in the pipe wall thus cooling the ceramic. The program provides a finite difference solution of the steady state conduction equation with proper boundary conditions in polar coordinates. The finite difference grid containing many irregular points because at the convective boundary conditions on the cold gas holes is prepared in a subroutine to allow running the program easily many times with different configurations.

## I - INTRODUCTION

For nuclear process heat applications high temperature reactors (HTR) must operate at a mean helium outlet temperature of 950°C or more. One of the most important technical problems associated with operation at such high temperatures is the transportation of helium from the core to the heat exchangers. Five possible HTR hot-gas duct designs have been reviewed by Kugeler et al.<sup>(1)</sup> as follows:

1. **Double walled piping with insulation in the annular space.** The outer steel pipe which carries the pressure load (40 bar) is kept at about 50°C. Fibres, metallic foil or ceramic can be used as insulation materials. The inner pipe which operates at a high temperature (950°C) does not carry any pressure load. In this design an inner liner for the hot gas is necessary and this creates problems because the performance of possible liner materials over a long service life has not at present been adequately tested. This type of duct with fibrous insulation in the annular space will be studied in the second test section of the IEA helium loop and a computer program has been developed<sup>(2)</sup> to predict its performance. The hot helium duct of the loop is of the same type as well.

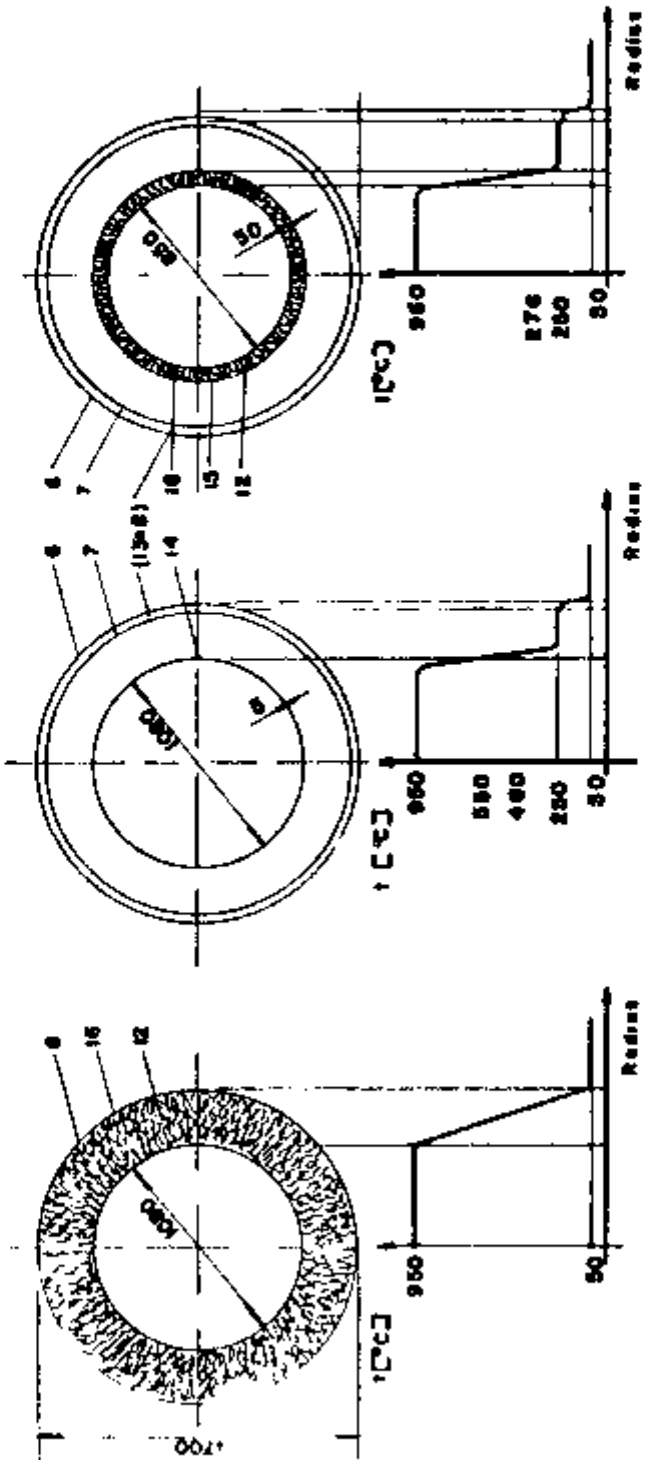
2. **Coaxial duct without insulation.** Countercurrent cold helium (250°C) flows in the annular duct between the inner and outer pipes. The outer wall again carries the pressure load and is maintained at about 250°C. The maximum inner wall temperature is limited to 550°C. The vibrations and thermal expansion are considered to be problematic in this duct design.

3. **Coaxial duct with insulated hot gas duct.** This is a combination of the two previous designs. It operates with an inner hot gas pipe insulated on the inside. Although the temperature of the inner wall is thus reduced, a liner which can withstand an operating temperature of 950°C is again necessary. The IEA helium loop presently has a test section to investigate this type of duct where the insulation consists of layers of metallic foil.

The dimensions and temperature profiles for the above hot gas ducts are shown in Figure 1 taken from reference<sup>(1)</sup>.

4. **Coaxial Ring Tubes Duct.** This design is the subject of this report. It consists of a coaxial duct (Figure 2) made completely from a ceramic material. The problem of inner vibrations and thermal expansion encountered in the previous designs are avoided in this one. The ceramic material would be carbon stone and the temperature gradients in carbon stone are high but can be tolerated<sup>(1)</sup>.

1 HOT GAS DUCT      2 COAXIAL DUCT WITHOUT INSULATION      3 COAXIAL DUCT WITH INSULATED HOT GAS DUCT



- 6 ARMOR TUBE
- 7 FLOW LIMITATION
- 8 SUPPORT TUBE FOR HOT GAS DUCT
- 12 HOT GAS LINER
- 13 COLD GAS DUCT
- 14 HOT GAS INLET
- 15 HOT GAS SUPPORT TUBE
- 16 INSULATION

Figure 1 - Hot gas ducts.



1 COAXIAL RING TUBES DUCT

2 COAXIAL RING HOLES DUCT

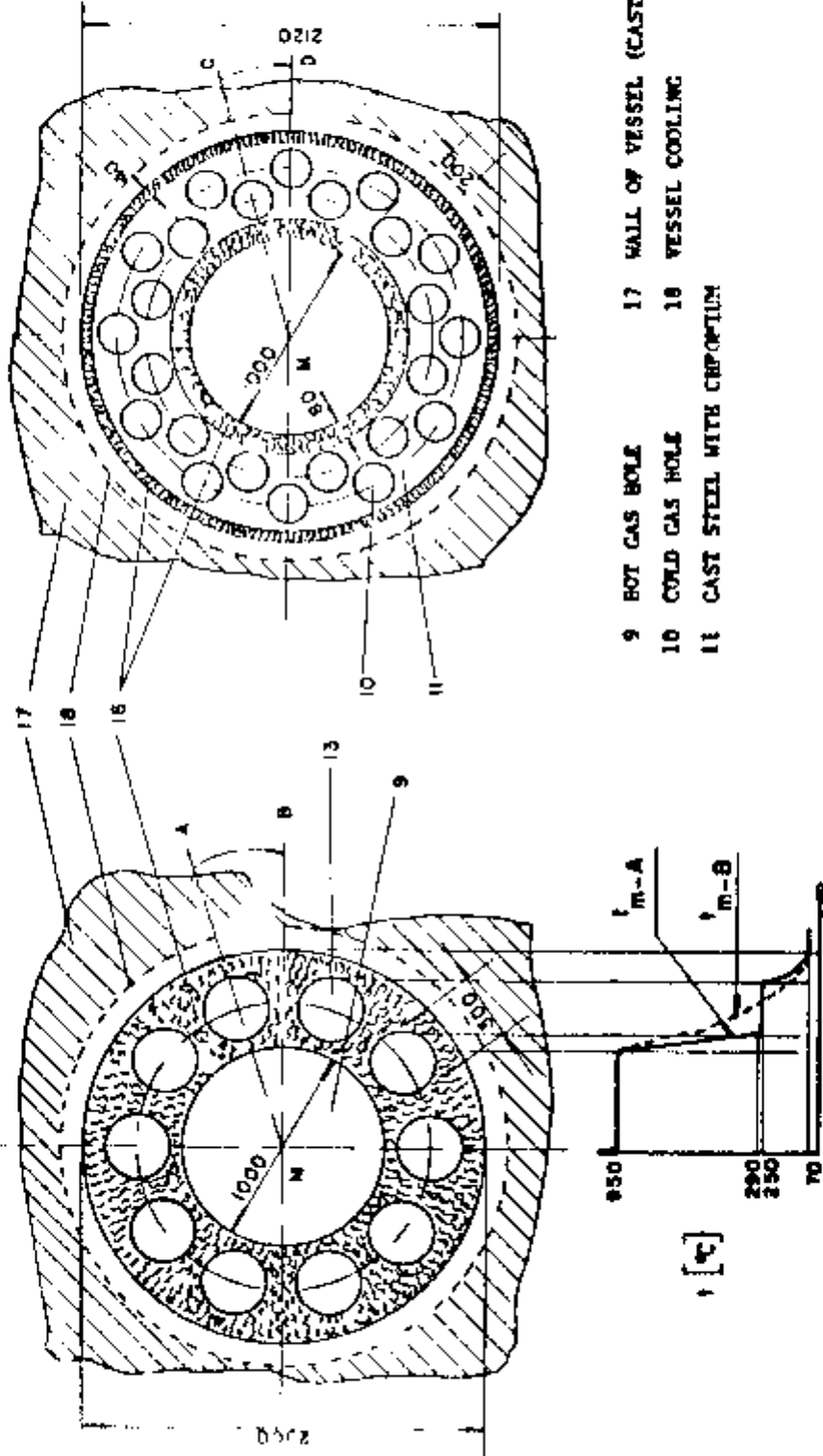


Figure 2 - Hot gas ducts

5 **Coaxial Ring Holes Duct** This design shown in Figure 2 has the cold gas ducting made of cast material with a carbon stone liner for the hot gas duct

The object of this report is to obtain the temperature distribution in a coaxial ring tubes duct made of ceramic material to help optimize the duct design. The resulting temperature distribution is necessary to calculate the thermal stresses in the ceramic material. The heat conduction equation in polar coordinates has been solved numerically using finite difference methods.

The parameters to be considered in the optimal design are the number of loops the HTR would have and the dimensions of the duct. The variables in the dimensions of the duct are the diameters of the hot gas duct and of the cold gas holes, the number of cold gas holes, the wall thickness of the ceramic material and the distance of the center of the cold gas hole to the center of the hot gas duct. To optimize the duct dimensions temperature profiles would have to be obtained for many different duct configurations. Since the finite difference grid contains many irregular points on and near the wall of the cold gas holes each configuration leads to a different grid. To avoid time consuming manual construction of the grid a subroutine has been written to prepare the grid configuration using only the duct dimensions as input. The output in the form of a table is used by the subroutine which mounts the coefficient matrix of the finite difference equations. Thus a computer program suitable for optimization studies has been obtained. The method of solution is discussed in detail in section II, and in section III some results are shown to illustrate the use of the program. Section IV contains the conclusions.

## II - FINITE DIFFERENCE EQUATIONS

### 1 - The Heat Conduction Equation and the Boundary Conditions

The steady state heat conduction equation in 2 dimensions (radial and angular) with no heat generation term is

$$\frac{\partial}{\partial r} \left( rK \frac{\partial T}{\partial r} \right) + \frac{1}{r} \frac{\partial}{\partial \theta} \left( K \frac{\partial T}{\partial \theta} \right) = 0 \quad (1)$$

with

- r radial coordinate
- $\theta$  angular coordinate
- T temperature
- K thermal conductivity = f(T)

The symmetry of the system allows modeling of the section shown in Figure 3 to obtain the temperature distribution in the hot gas duct considered.

The boundary conditions (Figure 3) for the section modeled are

$$r = r_1 \text{ (at the inside surface of the duct)}$$

$$K \frac{\partial T}{\partial r} + h_h (T_h - T) = 0 \quad (2)$$

$$r = r_2 \text{ (at the outside surface of the duct)}$$

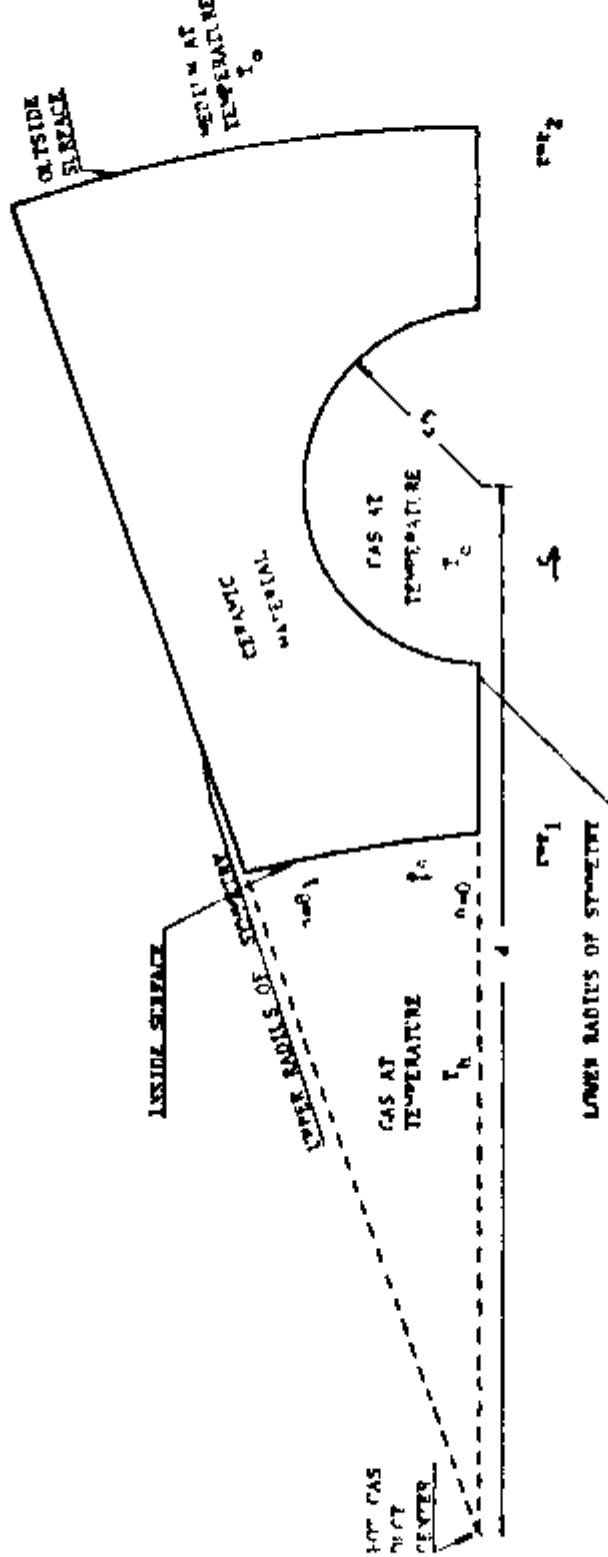


Figure 3 - Section modeled

$$K \frac{\partial T}{\partial r} - h_o (T_o - T) = 0 \quad (3-a)$$

Eq (3) is a convective boundary condition which really does not exist at the outside surface of the duct, but it is used to simulate either a prescribed outside wall temperature  $T_s$  or an insulated outside wall

Eq (3) is reduced to

$$\text{at } r = r_2 \quad T = T_s \quad (3-b)$$

by letting  $T_o = T_s$  and choosing a value for  $h_o$  which is much larger than  $K$

The boundary condition which corresponds to an insulated outside surface ( $\frac{\partial T}{\partial r}$  at surface = 0) is obtained from eq (3) by choosing  $h$  much smaller than  $K$ . Then eq (3) gives

$$\text{at } r = r_2 \quad \left( \frac{\partial T}{\partial r} \right) = 0 \quad (3-c)$$

$\phi = 0$  (lower diameter of symmetry) and  $\theta = \theta_1$  (upper diameter of symmetry)

$$K \frac{\partial T}{\partial \theta} = 0 \quad (4)$$

The boundary condition at the surface of the cold helium hole can not be readily expressed in terms of the coordinates  $r$  and  $\theta$ . Instead, it is written in terms of  $s$ , the normal to the surface

Thus on the surface of the cold helium hole

$$K \frac{\partial T}{\partial s} + h_c (T_c - T) = 0 \quad (5)$$

The terms in the boundary condition equations are defined as

$r_1$   $r_2$  internal and external radii of the duct respectively

$h_h$   $h_c$  heat transfer coefficients for the hot and cold gas respectively

$h_o$  heat transfer coefficient between the outer surface of the duct and the medium surrounding it

$T_h$   $T_c$  temperature of the hot and cold gas respectively

$T_o$  temperature of the medium surrounding the duct

$T_s$  temperature of the outside surface of the duct

$\theta_1$   $\pi/n$  where  $n$  is the number of cold gas holes

## 2 - Finite Difference Grid

A finite difference grid with 20 radial and 6 axial divisions is shown in Figure 4. The geometrical configuration of the grid is prepared in a subroutine whose inputs are the internal and external radii of the duct ( $r_1$  and  $r_2$ ), the radius ( $r_3$ ) of the cold gas hole, the distance from the center of the cold gas hole to the center of the duct ( $d$ ), the angle  $\theta_1$ , and the number of radial ( $m$ ) and angular ( $n$ ) divisions. The subroutine does the following (Table I):

- Numbers the grid points in a systematic way counting from left to right beginning on the lower radius of symmetry and going upward. Points on the cold gas hole and on a circle (such as points 43, 63, 75, etc) are numbered after points 42, 62 and 74.
- Determines the type of each point such as points on the cold gas hole (16, 28, 43, ...), on the hot gas duct surface (11, 23, ...) regular internal points (39, 77, ...) etc. Each type of point has a different finite difference equation.
- Identifies the neighbors of each point specifying the direction of each neighbor and its radial distance or angle to the point. For example, for point 27, points 28, 41, 26 and 15 are listed as its righthand, upper, lefthand and lower neighbors. For points on the cold gas hole such as point 28, the neighbors are points 27 and 41 because the radius of the cold gas hole passing through 28 intersects the nearest radius or circle (in this case a circle) between points 27 and 41 at point 28. The distance of point 28 to point 28 and the angular distance of point 28 to points 41 and 27 are calculated using trigonometric relations. For point 61, neighbors are points 60 and 59 because the radius of the cold gas duct passing through 61 cuts the nearest grid radius or circle (in this case a radius) between points 60 and 59. The distance between points 61 and 61 and the distance of point 61 to points 59 and 60 are calculated again using trigonometric relations.

Another finite grid example with 18 radial and 5 angular divisions is shown in Figure 4 and Table II.

## 3 - Finite Difference Equations

The finite difference form of eq. (1) is derived considering the different types of points of the finite difference grid.

a) **Regular internal points:** The finite difference equations are derived using central differences. At point  $p(i,j)$  where  $p(i,j)$  is a regular point, the finite difference form of eq. (1) is (see Figure 5):

$$\frac{1}{\Delta r^2} [r^{i+1/2} K^{i+1/2} (T^{i+1,j} - T^{i,j}) - r^{i-1/2} K^{i-1/2} (T^{i,j} - T^{i-1,j})] + \frac{1}{r^i \Delta \theta^2} [K^{i+1/2} (T^{i,j+1} - T^{i,j}) - K^{i-1/2} (T^{i,j} - T^{i,j-1})] = 0 \quad (6)$$

where

$$r^{i+1/2} = r^i + \frac{\Delta r}{2} \quad (7.1)$$

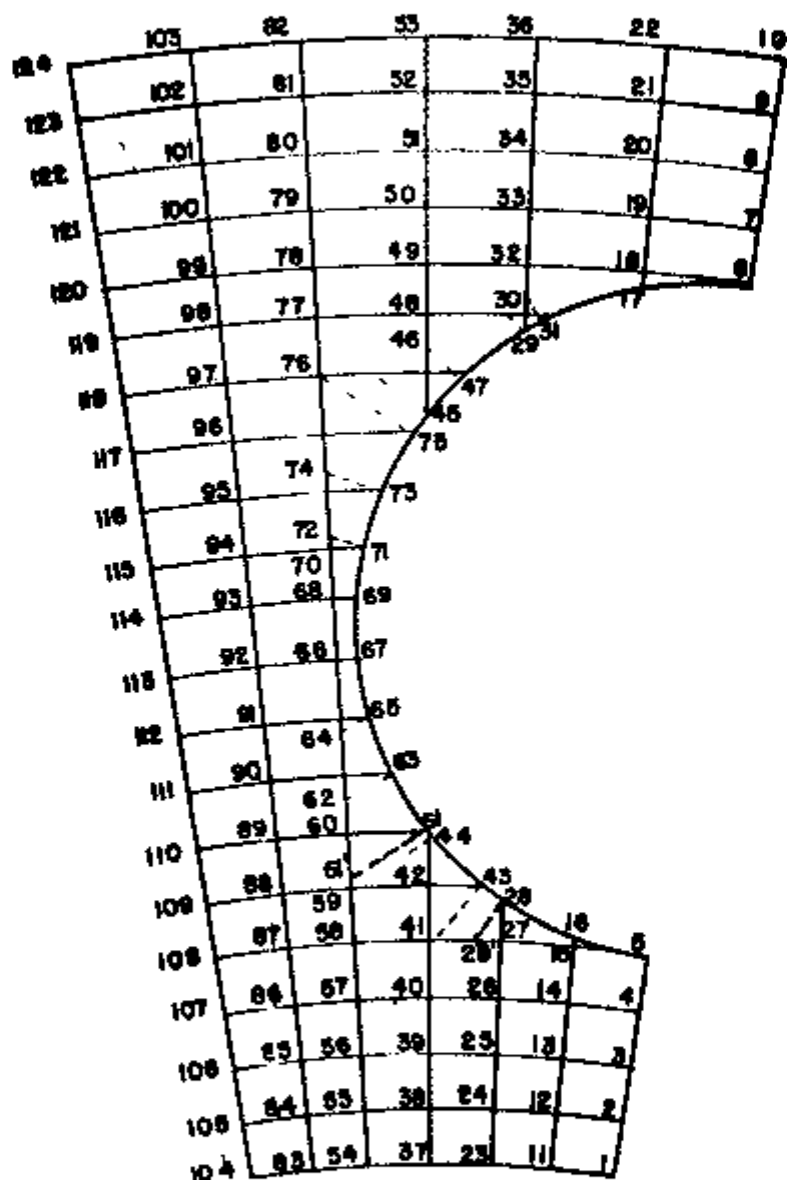


Figure 4 - Finite difference grid

$r_1 = 50$  cm  $r_2 = 100$  cm  $r_3 = 15$  cm  $d = 75$  cm

10 Cold gas holes

20 Radial and 6 angular divisions

Table I  
(Sheet 1 of 3)

## GRID CONFIGURATION

C1	C2	C3	C4	C5	C6	C7	C8	C9	C10	C11	C12						
1	6	2	11	И	0	2	ИИ	3	ИИИ	И	ИИИ	И	ИИИ	50	ИИИ	0	ИИИ
2	4	3	12	1	И	2	ИИ	3	ИИИ	1	ИИИ	0	ИИИ	52	ИИИ	0	ИИИ
3	4	4	13	2	И	2	ИИ	1	ИИИ	1	ИИ	И	ИИИ	53	ИИИ	0	ИИИ
4	4	5	14	3	0	1	ИИ	1	ИИИ	1	ИИ	И	ИИИ	54	ИИИ	0	ИИИ
5	17	0	14	И	4	И	ИИИ	3	ИИИИ	И	ИИИ	И	ИИИ	И	ИИИ	1	ИИИ
6	17	И	19	И	7	И	ИИИ	3	ИИИ	0	ИИИ	И	ИИИ	И	ИИИ	1	ИИИ
7	4	3	19	5	И	2	ИИ	1	ИИИ	2	ИИ	И	ИИИ	32	ИИИ	0	ИИИ
8	4	9	19	7	0	1	ИИ	3	ИИИ	1	ИИИ	0	ИИИ	35	ИИИ	И	ИИИ
9	4	10	11	8	И	2	ИИ	1	ИИИ	2	ИИ	0	ИИИ	37	ИИИ	0	ИИИ
10	8	И	12	9	0	И	ИИИ	1	ИИИ	2	ИИИ	И	ИИИ	100	ИИИ	0	ИИИ
11	2	12	11	0	1	1	ИИ	3	ИИИ	И	ИИИ	3	ИИИ	50	ИИИ	0	ИИИ
12	1	11	14	11	2	1	ИИ	3	ИИИ	2	ИИ	3	ИИИ	52	ИИ	И	ИИИ
13	1	14	13	11	3	1	ИИ	1	ИИИ	2	ИИ	3	ИИИ	55	ИИИ	0	ИИИ
14	1	15	13	13	4	1	ИИ	1	ИИИ	2	ИИИ	3	ИИИ	57	ИИИ	И	ИИИ
15	10	16	17	14	5	И	ИИИ	1	ИИИ	1	ИИ	3	ИИИ	61	ИИИ	0	ИИИ
16	17	0	17	0	15	0	ИИИИ	1	ИИ	И	ИИИ	0	ИИИ	И	ИИИ	И	437
17	17	0	31	0	15	0	ИИИИ	2	ИИ	И	ИИИ	0	ИИИ	И	ИИИ	0	610
18	11	19	32	17	0	1	ИИ	1	ИИИ	0	ИИИ	1	ИИИ	30	ИИИ	0	ИИИ
19	1	20	33	18	7	2	ИИ	3	ИИИ	2	ИИ	3	ИИИ	32	ИИ	0	ИИИ
20	1	21	34	19	0	2	ИИ	3	ИИИ	1	ИИ	3	ИИИ	35	ИИИ	0	ИИИ
21	1	22	35	20	9	2	ИИИ	1	ИИИ	2	ИИ	3	ИИИ	37	ИИ	0	ИИИ
22	3	0	36	21	10	И	ИИИ	1	ИИИ	И	ИИ	3	ИИИ	100	ИИИ	0	ИИИ
23	2	24	37	0	11	1	ИИ	1	ИИИ	И	ИИИ	3	ИИИ	50	ИИИ	И	ИИИ
24	1	25	38	23	11	2	ИИ	3	ИИИ	1	ИИИ	3	ИИИ	52	ИИИ	1	ИИИ
25	1	26	39	24	12	2	ИИИ	3	ИИИ	2	ИИ	3	ИИИ	55	ИИИ	1	ИИИ
26	1	27	40	25	11	2	ИИ	3	ИИИ	2	ИИИ	3	ИИИ	57	ИИИ	0	ИИИ
27	10	28	41	26	19	1	ИИ	3	ИИИ	2	ИИИ	3	ИИИ	61	ИИИ	0	ИИИ
28	17	0	41	0	27	0	ИИИ	1	ИИИ	И	ИИИ	1	ИИИ	0	ИИИ	2	125
29	17	0	48	0	30	И	ИИИ	2	ИИИ	0	ИИИ	И	ИИИ	И	ИИИ	0	145
30	16	32	48	29	31	2	ИИИ	1	ИИИ	И	123	И	133	97	500	0	000
31	18	32	0	30	И	155	И	ИИИ	И	342	0	ИИИ	0	000	0	399	
32	1	33	49	30	18	2	ИИ	3	ИИИ	2	ИИ	3	ИИИ	90	000	0	000
33	1	34	50	31	19	1	ИИ	3	ИИИ	2	ИИ	3	ИИИ	92	500	0	000
34	1	35	51	33	20	2	ИИ	3	ИИИ	1	ИИ	3	ИИИ	35	ИИИ	0	000
35	1	36	52	34	11	1	ИИ	3	ИИИ	1	ИИИ	3	ИИИ	97	500	0	000
36	3	0	53	35	21	И	ИИИ	3	ИИИ	2	ИИИ	3	ИИИ	100	000	0	000
37	2	38	54	0	23	2	ИИ	3	ИИИ	И	ИИИ	3	ИИИ	50	000	0	000
38	1	39	55	37	24	1	ИИ	3	ИИИ	2	ИИ	3	ИИИ	52	500	0	000
39	1	40	56	36	25	2	ИИ	3	ИИИ	2	ИИ	3	ИИИ	55	000	И	000
40	1	41	57	39	27	2	ИИ	3	ИИИ	2	ИИИ	3	ИИИ	57	500	0	000
41	1	42	58	40	27	1	ИИ	3	ИИИ	2	ИИИ	3	ИИИ	30	000	0	000
42	15	44	3	41	43	1	ИИ	3	ИИИ	1	ИИИ	2	ИИИ	32	500	0	000
43	18	И	41	0	17	И	ИИИ	И	ИИ	И	ИИИ	1	ИИИ	0	000	3	177
44	17	И	33	И	42	И	ИИИ	И	ИИ	И	ИИИ	1	ИИИ	И	ИИИ	3	684
45	17	И	31	И	35	И	ИИИ	1	ИИИ	И	ИИИ	1	ИИИ	И	ИИИ	2	446

Table I  
(Sheet 2 of 3)

11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	32
46	16	43	75	45	47	2	100	3	1000	1	577	0	371	27	1000	0	1000	0	1000	0	1000
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87	1	88	108	86	58	0	100	3	1000	0	500	3	1000	60	1000	0	1000	0	1000	0	1000
88	1	89	109	87	59	0	100	3	1000	0	500	3	1000	62	500	0	1000	0	1000	0	1000
89	1	90	110	88	60	1	100	3	1000	0	500	3	1000	65	1000	0	1000	0	1000	0	1000
90	1	91	111	89	61	1	100	3	1000	0	500	3	1000	67	500	0	1000	0	1000	0	1000



Table I  
(Sheet 3 of 3)

C1	C2	C3	C4	C5	C6	C7	C8	C9	C10	C11	C12
91	1	92	112	94	94	2	500	3	000	2	000
92	1	93	113	91	96	2	500	3	000	2	000
93	1	94	114	92	99	1	500	1	000	2	000
94	1	95	115	91	74	1	500	1	000	2	000
95	1	96	116	94	72	2	500	1	000	2	000
96	1	97	117	95	74	1	500	3	000	2	000
97	1	98	118	96	76	1	500	1	000	2	000
98	1	99	119	97	77	1	500	3	000	2	000
99	1	100	120	98	73	1	500	1	000	2	000
100	1	101	121	99	79	2	500	3	000	2	000
101	1	102	122	100	80	1	500	1	000	2	000
102	1	103	123	101	81	2	500	1	000	2	000
103	3	104	124	102	82	1	500	1	000	2	000
104	7	105	125	103	83	1	500	0	000	0	000
105	5	106	126	104	84	1	500	0	000	2	000
106	5	107	127	105	85	1	500	0	000	2	000
107	5	108	128	106	86	1	500	0	000	2	000
108	5	109	129	107	87	1	500	0	000	2	000
109	5	110	130	108	88	1	500	0	000	2	000
110	5	111	131	109	89	1	500	0	000	2	000
111	5	112	132	110	90	1	500	0	000	2	000
112	5	113	133	111	91	1	500	0	000	2	000
113	5	114	134	112	92	1	500	0	000	2	000
114	5	115	135	113	93	1	500	0	000	2	000
115	5	116	136	114	94	1	500	0	000	2	000
116	5	117	137	115	95	1	500	0	000	2	000
117	5	118	138	116	96	1	500	0	000	2	000
118	5	119	139	117	97	2	500	0	000	1	000
119	5	120	140	118	98	2	500	0	000	1	000
120	5	121	141	119	99	2	500	0	000	2	000
121	5	122	142	120	100	1	500	0	000	2	000
122	5	123	143	121	101	1	500	0	000	2	000
123	5	124	144	122	102	1	500	0	000	2	000
124	8	125	145	123	103	0	000	0	000	2	000

**DEFINITIONS** (Distances in cm angles in degrees)

- C1 Grid point identification number
- C2 Type of point
- C3 Identification number of the right hand neighbor
- C4 Identification number of the upper neighbor
- C5 Identification number of the left hand neighbor
- C6 Identification number of the lower neighbor
- C7 Distance between the point and the right hand neighbor
- C8 Angle between the point and the upper neighbor
- C9 Distance between the point and the left hand neighbor
- C10 Angle between the point and the lower neighbor
- C11 Radial coordinate ( $r'$ ) of the point
- C12 Distance  $\Delta s$  (Figures 7 A and 7 B)

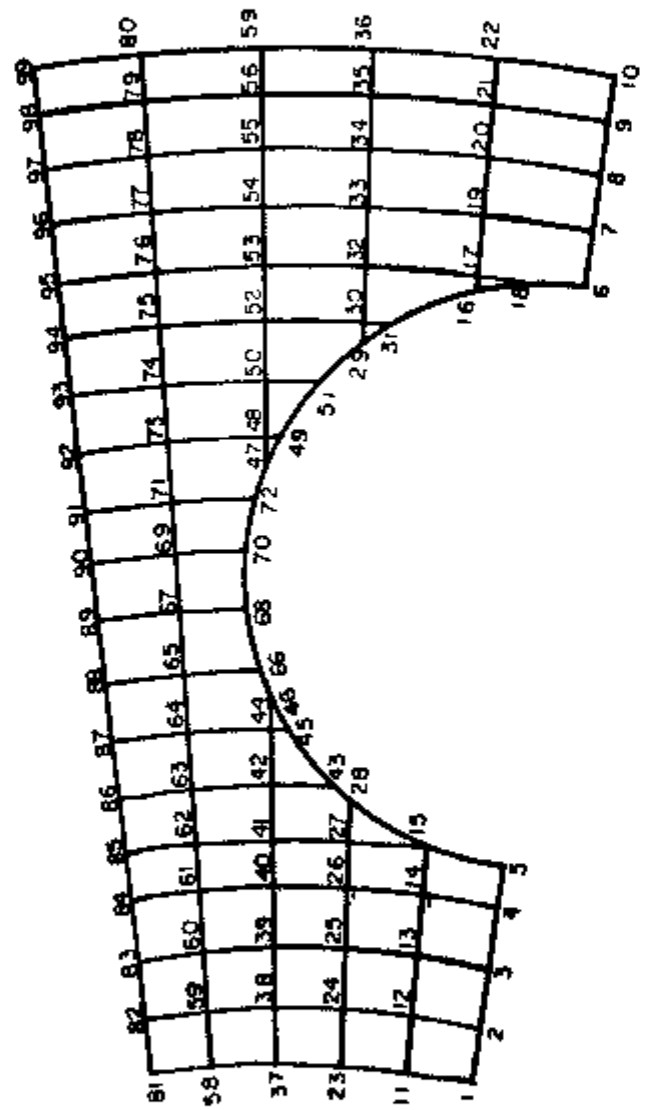


Figure 4 -- Elongation

Table II  
(Sheet 1 of 3)

GF1E CONFIGURATION

C1	C2	C3	C4	C5	C6	C7	C8	C9	C10	C11	C12						
1	6	2	11	0	0	2	500	3	000	0	000	52	500	0	000		
2	4	3	12	1	0	2	500	1	000	2	500	0	000	55	000	0	000
3	4	4	13	2	0	2	500	3	000	2	500	0	000	57	500	0	000
4	13	5	11	1	0	2	000	1	000	2	500	0	000	60	000	0	000
5	17	0	14	0	4	0	000	1	000	0	000	0	000	0	000	2	000
6	17	0	19	0	7	0	000	3	000	0	000	0	000	0	000	2	000
7	14	0	19	0	0	2	500	3	000	2	000	0	000	40	000	0	000
8	4	9	20	7	0	2	500	1	000	2	500	0	000	92	500	0	000
9	4	10	21	0	0	2	500	3	000	2	500	0	000	95	000	0	000
10	9	0	22	9	0	0	000	1	000	2	500	0	000	97	500	0	000
11	2	12	21	0	1	2	500	1	000	0	000	1	000	52	500	0	000
12	1	11	21	11	2	2	500	3	000	2	500	3	000	55	000	0	000
13	1	14	25	12	1	2	500	1	000	2	500	3	000	57	500	0	000
14	1	15	20	17	4	2	000	3	000	2	500	1	000	60	000	0	000
15	17	0	20	0	14	0	000	2	011	0	000	0	754	0	000	2	634
16	17	0	20	0	17	0	000	2	400	0	000	0	044	0	000	0	202
17	16	19	22	16	16	2	000	1	000	0	210	0	410	0	000	0	000
18	16	0	19	0	7	0	000	0	010	0	000	2	944	0	000	2	525
19	1	20	21	17	7	2	500	1	000	2	500	1	000	90	000	0	000
20	1	21	24	19	0	2	500	3	000	2	500	1	000	92	500	0	000
21	1	22	25	20	9	2	500	3	000	2	500	3	000	95	000	0	000
22	3	0	26	21	10	0	000	3	000	2	500	1	000	97	500	0	000
23	2	24	27	0	11	1	500	3	000	0	000	1	000	52	500	0	000
24	1	25	30	23	12	2	500	3	000	2	500	1	000	55	000	0	000
25	1	26	29	24	13	2	500	3	000	2	500	3	000	57	500	0	000
26	1	27	40	25	14	2	500	3	000	2	500	1	000	60	000	0	000
27	10	20	41	26	15	1	719	3	000	2	500	3	000	62	500	0	000
28	17	0	41	0	27	0	000	1	799	0	000	1	201	0	000	2	173
29	17	0	52	0	30	0	000	2	974	0	000	0	021	0	000	0	052
30	16	32	52	29	11	2	500	1	000	0	041	0	017	85	000	0	000
31	10	32	0	30	0	2	420	0	000	0	072	0	000	0	000	0	090
32	1	31	31	30	17	2	500	3	000	2	500	3	000	87	500	0	000
33	1	34	34	32	19	2	500	1	000	2	500	3	000	90	000	0	000
34	1	35	55	33	20	2	500	1	000	2	500	3	000	92	500	0	000
35	1	36	56	34	21	2	500	3	000	2	500	3	000	95	000	0	000
36	3	0	57	35	22	0	000	3	000	2	500	3	000	97	500	0	000
37	2	38	58	0	23	1	500	1	000	0	000	3	000	52	500	0	000
38	1	39	59	27	24	2	500	1	000	2	500	1	000	55	000	0	000
39	1	40	60	28	2	2	500	0	000	2	500	1	000	57	500	0	000
40	1	41	01	29	0	2	500	3	000	2	500	1	000	60	000	0	000
41	1	42	02	40	0	2	500	1	000	2	500	3	000	62	500	0	000
42	12	41	01	41	0	2	500	1	000	2	500	1	100	65	000	0	000
43	16	42	0	43	0	0	000	0	010	0	000	0	000	0	000	3	470
44	10	43	04	4	1	0	000	2	500	0	441	67	500	0	000	0	000
45	16	44	0	4	1	0	000	1	4	0	000	0	000	0	000	0	000

Table II  
(Sheet 2 of 3)

C1	C2	C3	C4	C5	C6	C7	C8	C9	C10	C11	C12						
46	17	0	64	0	44	0	000	1	200	0	000	2	349				
47	17	0	73	0	46	0	000	2	517	0	000	0	744				
48	16	50	70	47	49	2	500	3	000	0	125	0	000	0	000		
49	18	50	0	48	0	2	418	0	000	0	082	0	000	0	000	0	170
50	12	52	74	46	51	2	500	3	000	2	500	1	260	62	500	0	000
51	18	52	0	50	0	0	964	0	000	1	516	0	000	0	000	2	377
52	1	53	75	50	30	2	500	3	000	2	500	3	000	85	000	0	000
53	1	54	76	52	32	2	500	3	000	2	500	3	000	87	500	0	000
54	1	55	77	53	33	2	500	3	000	2	500	3	000	90	000	0	000
55	1	56	78	54	34	2	500	3	000	2	500	3	000	92	500	0	000
56	1	57	79	55	35	2	500	3	000	2	500	3	000	95	000	0	000
57	3	0	60	56	36	0	000	3	000	2	500	3	000	97	500	0	000
58	2	59	61	0	37	2	500	3	000	0	000	3	000	52	500	0	000
59	1	60	62	58	38	2	500	3	000	2	500	3	000	55	000	0	000
60	1	61	63	59	39	2	500	3	000	2	500	3	000	57	500	0	000
61	1	62	64	60	40	2	500	3	000	2	500	3	000	60	000	0	000
62	1	63	65	61	41	2	500	3	000	2	500	3	000	62	500	0	000
63	1	64	66	62	42	2	500	3	000	2	500	3	000	65	000	0	000
64	1	65	67	63	44	2	500	3	000	2	500	3	000	67	500	0	000
65	12	67	66	64	46	2	500	3	000	2	500	2	500	70	000	0	000
66	18	65	0	64	0	0	901	0	000	1	594	0	000	0	000	3	165
67	12	69	69	65	66	2	500	3	000	2	500	2	075	72	500	0	000
68	18	67	0	65	0	0	211	0	000	2	269	0	000	0	000	2	632
69	12	71	90	67	70	2	500	3	000	2	500	2	056	75	000	0	000
70	18	71	0	69	0	2	217	0	000	0	283	0	000	0	000	2	711
71	12	73	91	69	72	2	500	3	000	2	500	2	401	77	500	0	000
72	18	73	0	71	0	1	498	0	000	1	002	0	000	0	000	3	419
73	1	74	92	71	48	2	500	3	000	2	500	3	000	80	000	0	000
74	1	75	93	73	50	2	500	3	000	2	500	3	000	82	500	0	000
75	1	76	94	74	52	2	500	3	000	2	500	3	000	85	000	0	000
76	1	77	95	75	53	2	500	3	000	2	500	3	000	87	500	0	000
77	1	78	96	76	54	2	500	3	000	2	500	3	000	90	000	0	000
78	1	79	97	77	55	2	500	3	000	2	500	3	000	92	500	0	000
79	1	80	98	78	56	2	500	3	000	2	500	3	000	95	000	0	000
80	3	0	99	79	57	0	000	3	000	2	500	3	000	97	500	0	000
81	7	82	0	0	58	2	500	0	000	0	000	3	000	52	500	0	000
82	5	81	0	81	59	2	500	0	000	2	500	3	000	55	000	0	000
83	5	84	0	82	60	2	500	0	000	2	500	3	000	57	500	0	000
84	5	85	0	83	61	2	500	0	000	2	500	3	000	60	000	0	000
85	5	86	0	84	62	2	500	0	000	2	500	3	000	62	500	0	000
86	5	87	0	85	63	2	500	0	000	2	500	3	000	65	000	0	000
87	5	88	0	86	64	2	500	0	000	2	500	3	000	67	500	0	000
88	5	89	0	87	65	2	500	0	000	2	500	3	000	70	000	0	000
89	5	0	0	88	66	2	500	0	000	2	500	3	000	72	500	0	000
90	5	1	0	89	67	2	500	0	000	2	500	3	000	75	000	0	000

Table II  
(Sheet 3 of 3)

C1	C2	C3	C4	C5	C6	C7	C8	C9	C10	C11	C12
91	5	92	0	90	71	2 500	0 000	2 500	3 000	77 500	0 000
92	5	93	0	91	73	2 500	0 000	2 500	3 000	80 000	0 000
93	5	94	0	92	74	2 500	0 000	2 500	3 000	82 500	0 000
94	5	95	0	93	75	2 500	0 000	2 500	3 000	65 000	0 000
95	5	96	0	94	76	2 500	0 000	2 500	3 000	87 500	0 000
96	5	97	0	95	77	2 500	0 000	2 500	3 000	90 000	0 000
97	5	98	0	96	78	2 500	0 000	2 500	3 000	92 500	0 000
98	5	99	0	97	79	2 500	0 000	2 500	3 000	95 000	0 000
99	8	0	0	98	80	0 000	0 000	2 500	3 000	97 500	0 000

DEFINITION OF THE VARIABLES IS GIVEN IN TABLE - 1

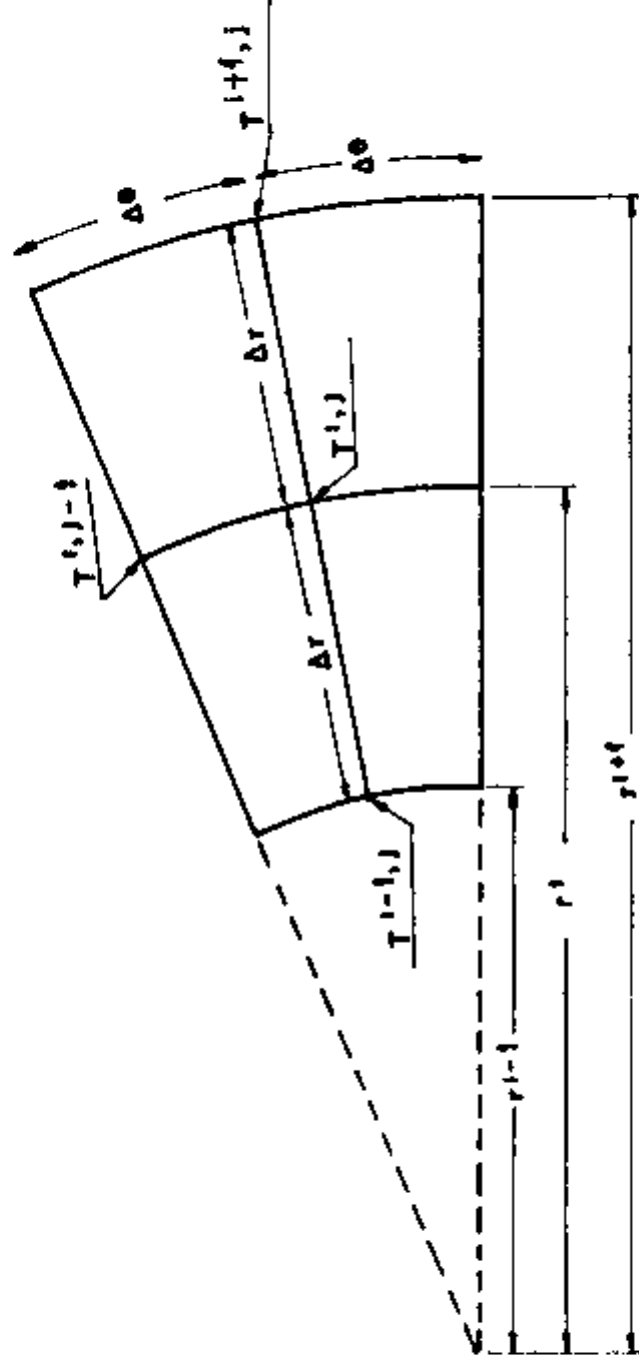


Figure 5 - Regular points.

$$r^{i-1/2} = r^i - \frac{\Delta r}{2} \quad (7.2)$$

$$K^{i+1/2} = \frac{K^{i+1} + K^i}{2} \quad (7.3)$$

$$K^{i-1/2} = \frac{K^{i-1} + K^i}{2} \quad (7.4)$$

$$K^{j+1/2} = \frac{K^{j+1} + K^j}{2} \quad (7.5)$$

$$K^{j-1/2} = \frac{K^{j-1} + K^j}{2} \quad (7.6)$$

Arranging eq (8) the coefficients of the 5 variables appearing in the finite difference equation at point  $p(i,j)$  are obtained as

$$T^{i-1} = \frac{r^{i-1/2} K^{i-1/2}}{\Delta r^2} \quad (8-1)$$

$$T^{i+1} = \frac{r^{i+1/2} K^{i+1/2}}{\Delta r^2} \quad (8-2)$$

$$T^{j-1} = \frac{K^{j-1/2}}{r^j \Delta \theta^2} \quad (8-3)$$

$$T^{j+1} = \frac{K^{j+1/2}}{r^j \Delta \theta^2} \quad (8-4)$$

$$T^i = - \frac{r^{i+1/2} K^{i+1/2} + r^{i-1/2} K^{i-1/2}}{\Delta r^2} - \frac{K^{j+1/2} + K^{j-1/2}}{r^i \Delta \theta^2} \quad (8-5)$$

Regular points are assigned type 1 (Table I)

b. Non regular internal points. Those are the points of type 10, 11, 12, 15 and 18 in Table I. The finite difference form of the heat conduction equation at point  $p(i,j)$  where  $p(i,j)$  is a non regular internal point is obtained by making a heat flow balance on the surface element shown in Figure 6. The heat balance gives

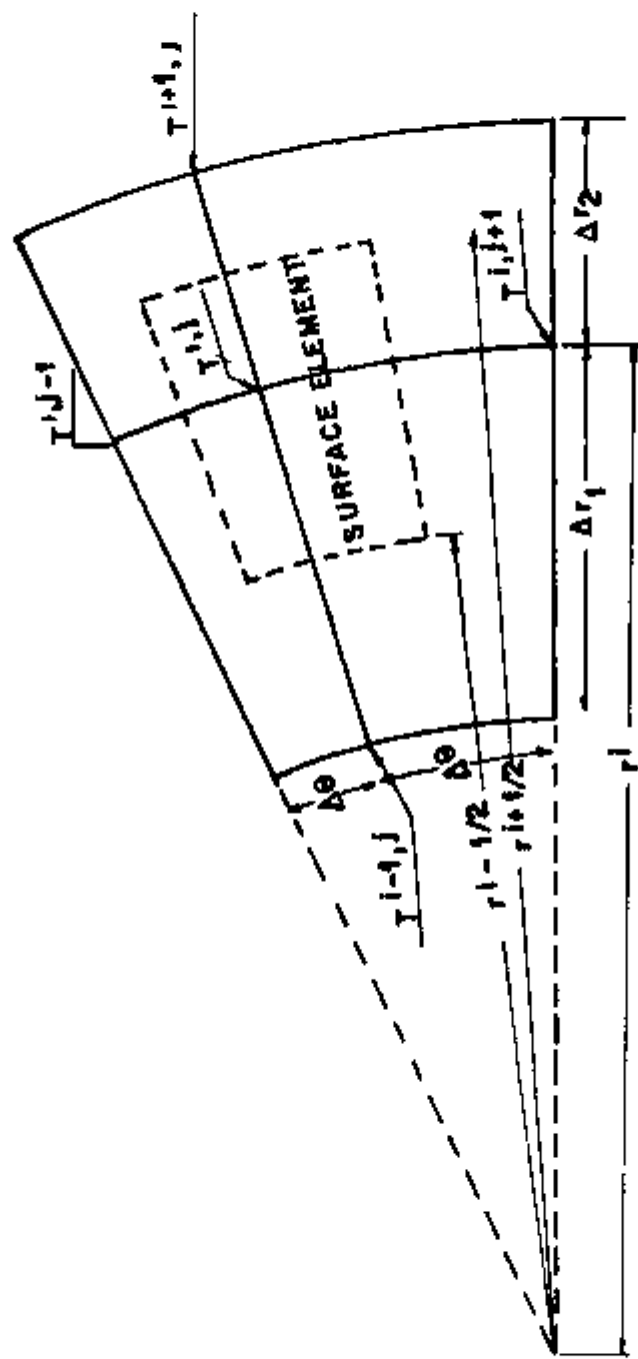


Figure 6 — Non regular internal points.



$$\begin{aligned} & \frac{\Delta\theta_1 + \Delta\theta_2}{2} [r^{i+1/2} K^{i+1/2} \left( \frac{\partial T}{\partial r} \right)^{i+1/2} - r^{i-1/2} K^{i-1/2} \left( \frac{\partial T}{\partial r} \right)^{i-1/2}] \\ & + \frac{\Delta r_1 + \Delta r_2}{2r^i} [K^{i+1/2} \left( \frac{\partial T}{\partial \theta} \right)^{i+1/2} - K^{i-1/2} \left( \frac{\partial T}{\partial \theta} \right)^{i-1/2}] = 0 \end{aligned} \quad (9)$$

Using central difference approximations for the derivatives

$$\left( \frac{\partial T}{\partial r} \right)^{i+1/2} = \frac{T^{i+1} - T^i}{\Delta r_2} \quad (10-1)$$

$$\left( \frac{\partial T}{\partial r} \right)^{i-1/2} = \frac{T^i - T^{i-1}}{\Delta r_1} \quad (10-2)$$

$$\left( \frac{\partial T}{\partial \theta} \right)^{i+1/2} = \frac{T^{i+1} - T^i}{\Delta \theta_2} \quad (10-3)$$

$$\left( \frac{\partial T}{\partial \theta} \right)^{i-1/2} = \frac{T^i - T^{i-1}}{\Delta \theta_1} \quad (10-4)$$

Combining eqs (9) to (10-4) and rearranging give the coefficients of the 5 variables appearing in the finite difference equation at point  $p(i)$

$$T^{i-1} \frac{r^{i-1/2} K^{i-1/2}}{\Delta r_1 (\Delta r_1 + \Delta r_2)} \quad (11-1)$$

$$T^{i+1} \frac{r^{i+1/2} K^{i+1/2}}{\Delta r_2 (\Delta r_1 + \Delta r_2)} \quad (11-2)$$

$$T^{i-1} \frac{K^{i-1}}{\Delta \theta_1 r^i (\Delta \theta_1 + \Delta \theta_2)} \quad (11-3)$$

$$T^{i+1} \frac{K^{i+1}}{\Delta \theta_2 r^i (\Delta \theta_1 + \Delta \theta_2)} \quad (11-4)$$

$$\begin{aligned} T^i & - \left( \frac{r^{i+1/2} K^{i+1/2}}{\Delta r_2 (\Delta r_1 + \Delta r_2)} + \frac{r^{i-1/2} K^{i-1/2}}{\Delta r_1 (\Delta r_1 + \Delta r_2)} \right) \\ & - \frac{1}{r^i (\Delta \theta_1 + \Delta \theta_2)} \left( \frac{K^{i+1/2}}{\Delta \theta_2} + \frac{K^{i-1/2}}{\Delta \theta_1} \right) \end{aligned} \quad (11-5)$$

Eqs (11-1) to (11-5) are reduced to eqs (8-1) to (8-5) when

$$\Delta r_1 = \Delta r_2 = \Delta r \quad \text{and} \quad \Delta \theta_1 = \Delta \theta_2 = \Delta \theta$$

c. **Points on the cold gas hole** In Table I those points are of type 17 when they are obtained by the intersection of a radius with the cold gas hole and of type 18 when they are obtained by the intersection of a circle with the cold gas hole

For non regular points  $r^{1+1/2}$  and  $r^{1-1/2}$  are defined as

$$r^{1+1/2} = \frac{r^{i+1} + r^i}{2}$$

$$r^{1-1/2} = \frac{r^i + r^{i-1}}{2}$$

In reference to Figure 4 for example point 28 is of type 17 and point 31 is of type 18

The finite difference form of eq (3) is obtained using a forward difference approximation for  $(\frac{\partial T}{\partial s})_s$  the temperature gradient at the surface of the cold gas hole. In reference to Figures 7-a and 7-b the forward difference approximation for  $(\frac{\partial T}{\partial s})_s$  is given by

$$\left(\frac{\partial T}{\partial s}\right)_s = \frac{T_a - T_s}{\Delta s} \quad (12)$$

where  $\Delta s$  is the distance between points (s) and (a) and  $T_s$  and  $T_a$  are the temperatures at points (s) and (a) respectively

Point (a) is obtained by the intersection of the cold gas hole radius passing through (s) with a circle or a radius. In Figure 7-a the radius passing through (s) first intersects a circle at point (a) and then intersect a radius at (a'). Since (s) is closer to (a) than to (a')  $\Delta s$  in eq (12) is taken as the distance of (s) to (a). Using the minimum possible  $\Delta r$  increases the accuracy of forward difference approximation for  $(\frac{\partial T}{\partial s})_s$ . In Figure 7-b the radius passing through (s) first intersects a radius at point (a), thus  $\Delta s$  is chosen as the distance of (s) to (a)

The temperature  $T_a$  in eq (12) is expressed in terms of temperatures  $T_1$  and  $T_2$  (Figures 7-a and 7-b) interpolating linearly between  $T_1$  and  $T_2$ . Thus

$$T_a = \Delta \theta_2 T_1 + \Delta \theta_1 T_2 \quad (13.1)$$

or

$$T_a = \Delta r_2 T_1 + \Delta r_1 T_2 \quad (13.2)$$

where  $\Delta r_1$ ,  $\Delta r_2$ ,  $\Delta \theta_1$  and  $\Delta \theta_2$  are defined in Figures 7-a and 7-b

The finite difference form of eq (3) thus becomes

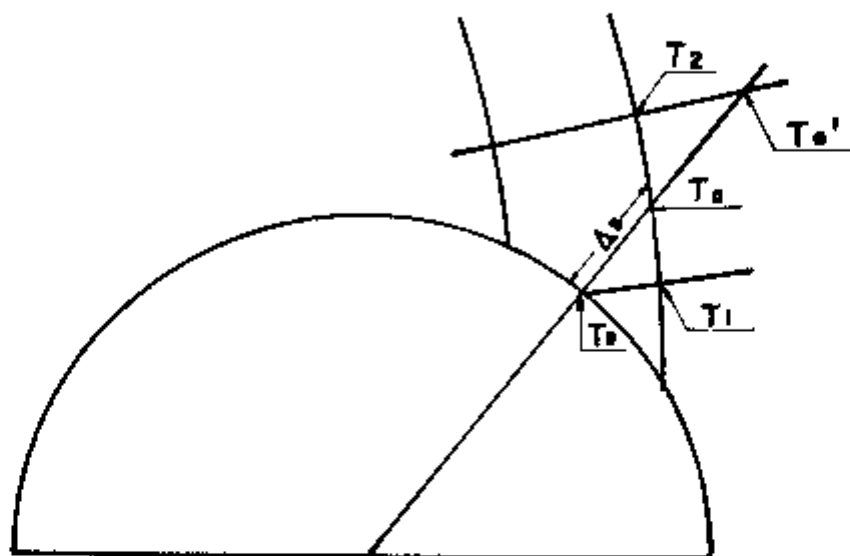


Figure 7 A - Points on the cold gas hole

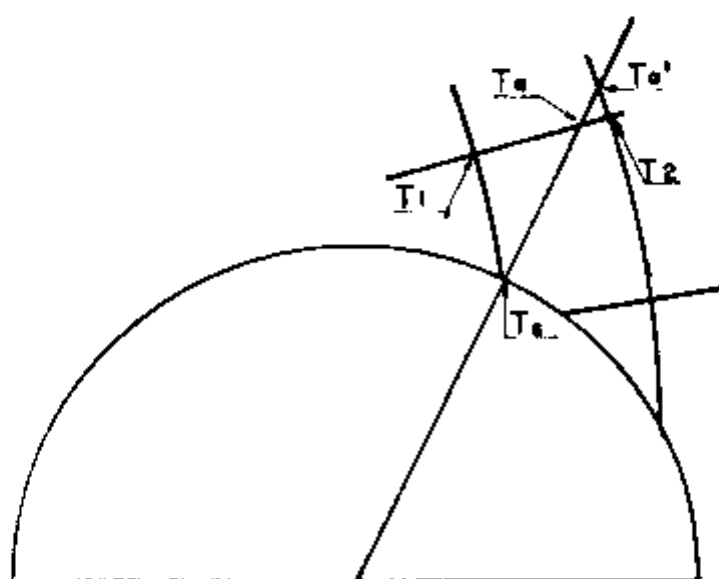


Figure 7 B - Points on the cold gas hole

$$\frac{K_s}{\Delta s} \left( \frac{\Delta\theta_2 T_1 + \Delta\theta_1 T_2}{\Delta\theta_1 + \Delta\theta_2} - T_s \right) + h_c (T_c - T_s) = 0 \quad (14-1)$$

or

$$\frac{K_s}{\Delta s} \left( \frac{\Delta r_2 T_1 + \Delta r_1 T_2}{\Delta r_1 + \Delta r_2} - T_s \right) + h_c (T_c - T_s) = 0 \quad (14-2)$$

The coefficients of the variables in eq (14-1) or (14-2) is then

$$T_1 = \frac{K_s \Delta\theta_2}{\Delta s (\Delta\theta_1 + \Delta\theta_2)} \quad (15-1a)$$

$$T_2 = \frac{K_s \Delta\theta_1}{\Delta s (\Delta\theta_1 + \Delta\theta_2)} \quad (15-2a)$$

$$T_s = - \left( \frac{K_s}{\Delta s} + h_c \right) \quad (15-3a)$$

or

$$T_1 = \frac{K_s \Delta r_2}{\Delta s (\Delta r_1 + \Delta r_2)} \quad (15-1b)$$

$$T_2 = \frac{K_s \Delta r_1}{\Delta s (\Delta r_1 + \Delta r_2)} \quad (15-2b)$$

$$T_s = - \left( \frac{K_s}{\Delta s} + h_c \right) \quad (15-3)$$

The right hand side B is

$$B = - h_c T_c \quad (15-4)$$

d Points on the lower diameter of symmetry In reference to Figure 4 those are points 105 to 123 and in Table I they can be identified as of type 4

The finite difference form of eq (1) is written at point (i, t) as follows (Figure B)

$$\begin{aligned} & \frac{1}{\Delta r^2} \left[ r^{i+1/2} K^{i+1/2, 1} (T^{i+1, 1} - T^{i, 1}) - r^{i-1/2} K^{i-1/2, 1} (T^{i, 1} - T^{i-1, 1}) \right] \\ & + \frac{1}{r^i \Delta\theta^2} \left[ K^{i, 3/2} (T^{i, 2} - T^{i, 1}) - K^{i, 1/2} (T^{i, 1} - T^{i, 0}) \right] = 0 \end{aligned} \quad (16)$$

Eq (16) can be obtained by setting  $F=1$  in the finite difference equation for regular points (eq (6))

The boundary condition equation on the lower diameter of symmetry that is at  $\theta = 0$

$$\frac{\partial T}{\partial \theta} = 0 \quad (4)$$

is used to write that at  $\theta = 0$

$$0 = \frac{\partial T}{\partial \theta} = \frac{T^{i,2} - T^{i,0}}{2\Delta\theta}$$

which gives

$$T^{i,0} = T^{i,2} \quad (17.1)$$

which could also be deduced directly from symmetry considerations

Eq (17.1) indicates that

$$T^{i,1/2} = T^{i,3/2} \quad (17.2)$$

and since thermal conductivity  $K$  is a function of temperature

$$K^{i,1/2} = K^{i,3/2} \quad (17.3)$$

Combining eqs (15.4) (17.1) and (17.3) and then rearranging eq (16) one obtains the coefficients of the temperatures  $T^{i-1,1}$ ,  $T^{i+1,1}$ ,  $T^{i,1}$  and  $T^{i,2}$  in the finite difference equation written at points on the lower diameter of symmetry

$$T^{i-1,1} = \frac{1}{\Delta r^2} (r^{i-1/2} K^{i-1/2,1}) \quad (18.1)$$

$$T^{i+1,1} = \frac{1}{\Delta r^2} (r^{i+1/2} K^{i+1/2,1}) \quad (18.2)$$

$$T^{i,2} = \frac{2K^{i,3/2}}{r^i \Delta \theta^2} \quad (18.3)$$

$$T^{i,1} = -\frac{1}{\Delta r^2} (r^{i+1/2} K^{i+1/2,1} + r^{i-1/2} K^{i-1/2,1}) - \frac{2K^{i,3/2}}{r^i \Delta \theta^2} \quad (18.4)$$

e Points on the upper diameter of symmetry In Figure 4 those are points 2 to 4 and 7 to 9 and in Table I they appear as of Type 5 The finite difference form of eq (1) is written at point (i,j) as follows (Figure 8)

$$\begin{aligned} & \frac{1}{\Delta r^2} [ r^{j+1/2} K^{j+1/2} (T^{j+1} - T^j) - r^{j-1/2} K^{j-1/2} (T^j - T^{j-1}) ] \\ & + \frac{1}{r^j \Delta \theta^2} [ K^{j+1/2} (T^{j+1} - T^j) - K^{j-1/2} (T^j - T^{j-1}) ] = 0 \end{aligned} \quad (19)$$

Eq (19) can be obtained from eq (8), setting  $j = 0$

The boundary condition on the upper diameter of symmetry, that is, at  $\theta = \theta_t$ ,

$$\left( \frac{\partial T}{\partial \theta} \right) = 0 \quad (3)$$

is used to write that at  $\theta = \theta_t$ ,

$$0 = \frac{\partial T}{\partial \theta} = \frac{T^{j+1} - T^{j-1}}{2\Delta \theta}$$

which gives

$$T^{j+1} = T^{j-1} \quad (20-1)$$

which could also be deduced directly from symmetry considerations

Eq (20-1) indicates that

$$T^{j+1/2} = T^{j-1/2} \quad (20-2)$$

or

$$K^{j+1/2} = K^{j-1/2} \quad (20-3)$$

Combining eqs (19), (20-1) and (20-3) and then rearranging eq (19) one obtains the coefficients of the temperatures  $T^{j-1}$ ,  $T^{j+1}$ ,  $T^j$  and  $T^{j-1}$  in the finite difference equation written at points on the upper diameter of symmetry

$$T^{j-1} \left[ \frac{1}{\Delta r^2} (r^{j-1/2} K^{j-1/2}) \right] \quad (21-1)$$

$$T^{j+1} \left[ \frac{1}{\Delta r^2} (r^{j+1/2} K^{j+1/2}) \right] \quad (21-2)$$

$$T^j \left[ \frac{2K^{j+1/2}}{r^j \Delta \theta^2} \right] \quad (21-3)$$

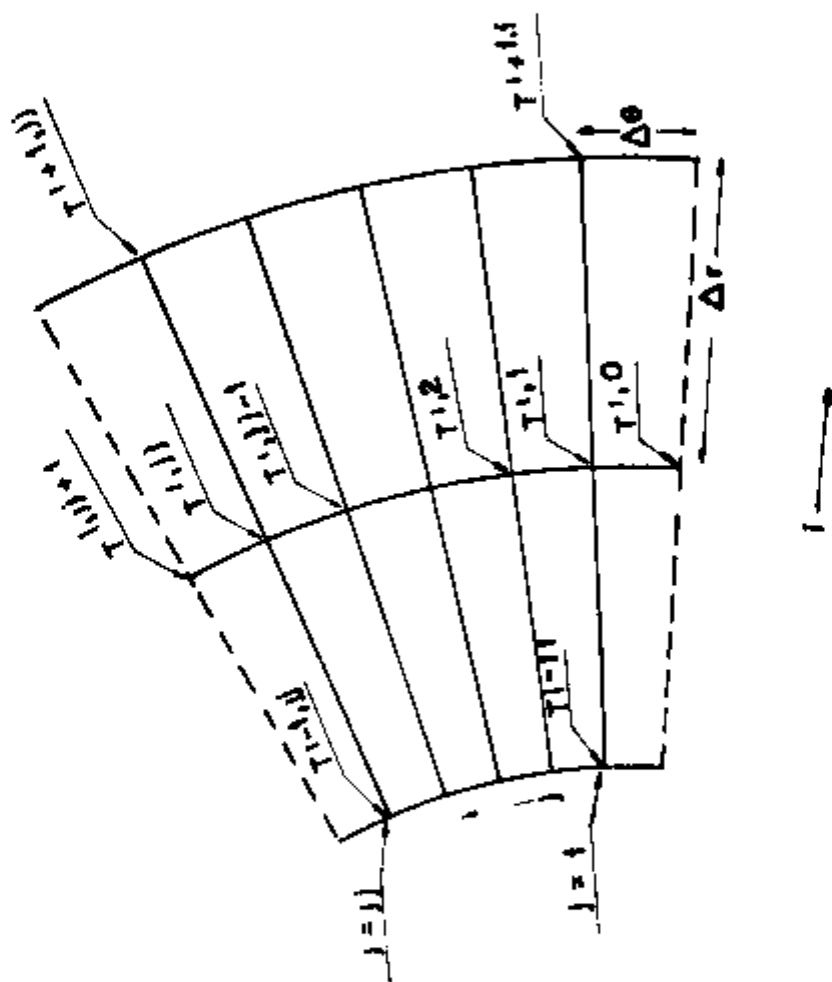


Figure 8 — Points on the diameters of symmetry

$$T^{1,j} = \frac{1}{\Delta r^2} (r^{j+1/2} K^{j+1/2,j} + r^{j-1/2} K^{j-1/2,j}) - \frac{2K^{j,j-1/2}}{r^j \Delta \theta^2} \quad (21.4)$$

f Points on the inside (hot) surface of the duct. These are points 1, 11, 23, 37, 54, 83 and 104 in Figure 4 and they are of type 2 in Table I except for point 1 which is of type 6 and point 104 which is of type 7.

The boundary condition given by eq. (2) is written in finite difference form as (Figure 8)

$$h_h (T_h - T^{1,j}) + \frac{K^{1,j}}{\Delta r} (T^{2,j} - T^{1,j}) = 0 \quad (22)$$

The coefficients of the temperatures  $T^{2,j}$  and  $T^{1,j}$  in eq. (22) are then

$$T^{2,j} = \frac{K^{1,j}}{r} \quad (23.1)$$

$$T^{1,j} = h_h - \frac{K^{1,j}}{\Delta r} \quad (23.2)$$

The right hand side of eq. (22) B is

$$B = -h_h T_h \quad (23.3)$$

g Points on the outside (cold) surface of the duct. These are points 8, 22, 38, 53, 82, 103 and 124 in Figure 4 and they are of type 3 in Table I except for point 38 which is of type 8 and point 124 which is of type 9.

The boundary condition given by eq. (3 a) is written in finite difference form as (Figure 9)

$$h_c (T_c - T^{ii,j}) + \frac{K^{ii,j}}{\Delta r} (T^{ii-1,j} - T^{ii,j}) = 0 \quad (24)$$

The coefficients of temperatures  $T^{ii-1,j}$  and  $T^{ii,j}$  in eq. (24) are

$$T^{ii-1,j} = \frac{K^{ii,j}}{\Delta r} \quad (25.1)$$

$$T^{ii,j} = h_c - \frac{K^{ii,j}}{\Delta r} \quad (25.2)$$

The right hand side of eq. (24) is

$$B = -h_c T_c \quad (25.3)$$





### b. Points on the lower diameter of symmetry next to the cold gas hole

These are points 4 and 7 in Figure 4. point 4 is assigned type 13 and point 7 type 14 in Table II. Note that points of type 13 or 14 do not exist in Figure 4 nor in Table I because the distance of the points 4 and 7 in Figure 4 to the cold gas hole is equal to  $\Delta r$ , therefore both are assigned type 4.

The finite difference form of equation (1) at point (i) on the left is obtained by considering it as a non-regular interior point. Then, a heat flow balance on the surface element shown in Figure 9 gives

$$\begin{aligned} \Delta \theta \left[ r^{i+1/2} K^{i+1/2} \left( \frac{\partial T}{\partial r} \right)^{i+1/2} - r^{i-1/2} K^{i-1/2} \left( \frac{\partial T}{\partial r} \right)^{i-1/2} \right] \\ + \frac{\Delta r + \Delta p}{2r^i} \left[ K^{i3/2} \left( \frac{\partial T}{\partial \theta} \right)^{i3/2} - K^{i1/2} \left( \frac{\partial T}{\partial \theta} \right)^{i1/2} \right] = 0 \end{aligned} \quad (26)$$

Using central difference approximations for the derivatives one obtains

$$\left( \frac{\partial T}{\partial r} \right)^{i+1/2} = \frac{T^{i+1} - T^i}{\Delta p} \quad (27-1)$$

$$\left( \frac{\partial T}{\partial r} \right)^{i-1/2} = \frac{T^i - T^{i-1}}{\Delta r} \quad (27-2)$$

$$\left( \frac{\partial T}{\partial \theta} \right)^{i3/2} = \frac{T^{i2} - T^i}{\Delta \theta} \quad (27-3)$$

$$\left( \frac{\partial T}{\partial \theta} \right)^{i1/2} = \frac{T^i - T^{i0}}{\Delta \theta} \quad (27-4)$$

Combining eqs (26) to (27-4) and noting that

$$T^{i0} = T^i \quad (17-1)$$

$$K^{i1/2} = K^{i3/2} \quad (17-3)$$

yields the coefficients of the temperatures  $T^{i-1}$ ,  $T^{i+1}$ ,  $T^{i2}$  and  $T^i$  in the finite difference equation written at the point on the left of the cold gas hole

$$T^{i-1} = \frac{r^{i-1/2} K^{i-1/2}}{\Delta r (\Delta r + \Delta p)} \quad (28-1)$$

$$T^{i+1} = \frac{r^{i+1/2} K^{i+1/2}}{\Delta p (\Delta r + \Delta p)} \quad (28-1)$$

$$T^{i,2} = \frac{K^{i,3/2}}{r^i \Delta\theta^2} \quad (28-3)$$

$$T^{i,1} = \frac{1}{(\Delta r + \Delta p)} \left( \frac{r^{i+1/2} K^{i+1/2,1}}{\Delta p} + \frac{r^{i-1/2} K^{i-1/2,1}}{\Delta r} \right) - \frac{K^{i,3/2}}{r^i \Delta\theta^2} \quad (28-4)$$

The coefficients of the temperatures in the finite difference equation written at point on the right of the cold gas hole are obtained in a similar way

$$T^{i-1,1} = \frac{r^{i-1/2} K^{i-1/2,1}}{\Delta p (\Delta r + \Delta p)} \quad (28-1)$$

$$T^{i+1,1} = \frac{r^{i+1} K^{i+1/2,1}}{\Delta r (\Delta r + \Delta p)} \quad (28-2)$$

$$T^{i,2} = \frac{K^{i,3/2}}{r^i \Delta\theta^2} \quad (28-3)$$

$$T^{i,1} = \frac{1}{\Delta r + \Delta p} \left( \frac{r^{i+1/2} K^{i+1/2,1}}{\Delta r} + \frac{r^{i-1/2} K^{i-1/2,1}}{\Delta p} \right) - \frac{K^{i,3/2}}{\Delta r^i \Delta\theta^2} \quad (28-4)$$

#### 4 - Numerical Solution

The finite difference form of the heat conduction equation — eq (1) and the boundary condition equations lead to a set of  $n$  algebraic equations where  $n$  is the number of unknown temperatures. The  $n$  algebraic equations must be solved simultaneously in order to obtain the unknown temperatures. The system of algebraic equations is linear when the thermal conductivity  $K$  is constant, and nonlinear when  $K$  is a function of the temperature. The nonlinear system is solved by the successive solutions of the linearized system. Values of  $K^{i,1}$  are evaluated at the arithmetic mean of the hot and cold gases for the first iteration and at the previously computed values of  $T^{i,1}$  for the subsequent iterations. The procedure is continued until all temperatures converged within a given error criterion.

Since the set of algebraic equations to be solved is sparse (in no equation appears more than 6 unknowns) subroutine SPAMAT has been used for the solution. SPAMAT solves a set of linear algebraic equations by Gauss elimination using sparse matrix techniques. Only the non zero elements of the coefficient matrix are considered for the storage and the elimination operations. Thus significant savings in computer memory and running time are achieved. SPAMAT is described in an other report<sup>(3)</sup>. The computer program is listed in Appendix B. It was run on a PDP 11 and took less than 2 minutes to run a problem with 124 grid points.

### III - RESULTS

The computer program has been run for 12 cases as shown in Table III. The dimensions of the different hot duct geometries used are given in Table IV. Geometry 1 is the one shown in Figure 2. Geometries 2 and 3 are obtained by multiplying the dimensions of geometry 1 by 1.1 and 0 respectively. The finite difference grid for geometries 1, 2, and 3 is given in Figure 4 and Table I.

Table III

## Summary of Runs

 $T_h = 950^\circ\text{C}$ 
 $T_c = 250^\circ\text{C}$ 
 for All Runs

$$\text{Units } K \frac{W}{m^2 \cdot ^\circ\text{C}} \quad v_h \frac{v_c}{s} \quad \frac{m}{s} \quad h_h \frac{h_c}{s} \quad h_o \frac{w}{m^2 \cdot ^\circ\text{C}} \quad T_o \frac{^\circ\text{C}}{m} \quad \Delta T/m \quad \frac{^\circ\text{C}}{m}$$

Run	Geometry	K	$v_h$	$v_c$	$h_h$	$h_c$	$h_o$	$T_o$	$\Delta T/m$	Table	Figure
1	G1 6 Loops	5.2	112	53	1333	1646	20000	70	128	A 1	10
2	G1 8 Loops	5.2	75	35	964	1190	20000	70	188	A 2	-
3	G1 12 Loops	5.2	58	26.5	765	945	20000	70	278	A 3	-
4	G1 6 Loops	118	112	53	1333	1646	20000	70	1.40	A-4	11
5	G1 9 Loops	118	75	35	964	1190	20000	70	1.75	A-5	-
6	G1 12 Loops	118	66	28.5	765	945	20000	70	2.04	A-6	-
7	G1 6 Loops	118 05T	112	53	1333	1646	20000	70	1.19	A 7	-
8	G2 9 Loops	5.2	82	28	812	1002	20000	70	187	A-8	-
9	G3 12 Loops	5.2	69	33	825	1142	20000	70	243	A-9	-
10	G1 6 Loops	5.2	112	53	1333	1646	140	25	128	A 10	10
11	G1 8 Loops	5.2	112	53	1333	1646	1	25	128	A 11	12
12	G4 6 Loops	5.2	101	59	1223	1841	140	25	143	A 12	13

Table IV

## Duct Dimension

	Geometry 1 (G1)	Geometry 2 (G2)	Geometry 3 (G3)	Geometry 4 (G4)
Number of Cold Gas Holes (N)	10	10	10	12
$r_1$ (m)	5	55	45	.525
$r_2$ (m)	1	1.1	90	975
$r_3$ (m)	15	165	135	13
$d$ (m)	75	825	675	75
$A_h$ (m <sup>2</sup> )	785	950	636	865
$A_c$ (m <sup>2</sup> )	707	855	573	637

$r_1$   $r_2$  Internal and external radii of the duct respectively

$r_3$  Radius of the cold gas hole

$d$  Hot gas duct - cold gas hole center to center distance

$A_h$  Area of the hot gas duct  $A_h = \pi r_2^2$

$A_c$  Cold gas hole flow area  $A_c = N \pi r_3^2$

Geometry 4 has a relatively smaller wall thickness than the previous ones. Thus it has a smaller cold gas hole diameter and more (12 instead of 10) cold gas holes. The finite difference grid for geometry 4 is shown in Figure 4 and Table II.

For the thermal conductivity of the ceramic material specified as carbon stone in (1) 3 values have been used. The first two are  $K = 5.2 \text{ Wm}/(\text{m}^2 \text{ } ^\circ\text{C})$  and  $K = 118 \text{ Wm}/(\text{m}^2 \text{ } ^\circ\text{C})$  corresponding to carbon pipe and graphite pipe at room temperature<sup>(4)</sup>.  $K$  for carbon pipe varies very little with temperature<sup>(4)</sup>, but the thermal conductivity of miscellaneous graphite as given in<sup>(5)</sup> ranges from 20 to 200  $\text{Wm}/(\text{m}^2 \text{ } ^\circ\text{C})$  in the range of 100 to 1000 $^\circ\text{C}$ . The third expression for  $K$  of the duct material has been taken as  $K = 118.05T$  ( $T$  in  $^\circ\text{C}$ ) to test the computer program when  $K$  varies with temperature.

In order to have tolerable pressure losses and dimensions in the design a hot gas velocity of about 60-70m/sec and a hot gas duct diameter of 1.11 in should not be exceeded<sup>(11)</sup>. According to Figure 8 in reference<sup>(11)</sup> these data lead to 8-12 loops for a 3000 MW plant. The computer program has been run for a plant with 8, 9 or 12 loops. The 6 loop case leads to highest velocities therefore to highest heat transfer coefficient on the hot gas side resulting in highest duct wall temperatures. Although unacceptable high velocities are thus reached the case was studied as an example of severe conditions.

The respective hot and cold gas temperatures have been taken as 950 $^\circ\text{C}$  and 250 $^\circ\text{C}$ . The outside duct temperature  $T_2$  is indicated as 70 $^\circ\text{C}$  in Figure 7 of reference<sup>(11)</sup>. Except for runs 10, 11 and 12 values of  $T_2$  near 70 $^\circ\text{C}$  have been obtained by using a relatively high value for  $h_o$  and taking the surrounding temperature  $T_o$  as 70 $^\circ\text{C}$ . In run 10 the surrounding temperature  $T_o$  has been taken as 25 $^\circ\text{C}$  and several values  $h_o$  have been tried until computed  $T_2$  was near 70 $^\circ\text{C}$ . As seen in Table A 10 when  $h_o = 140 \text{ W}/(\text{m}^2 \text{ } ^\circ\text{C})$   $T_2$  varies from 79.0 to 59.9 along the wall. When the effective thermal conductivity of the outer insulation is known  $h_o$  can be estimated and this estimated value can be used in the computer program as a realistic boundary condition. The computed temperature distribution will give the outside duct temperature  $T_o$ .

The heat transfer coefficients for the hot and cold gases ( $h_h$  and  $h_c$ ) have been calculated from

$$h = 0.23 K/D \text{ Re}^{0.8} \text{ Pr}^{0.4} \quad (30)$$

The helium properties have been taken from<sup>(6)</sup>

The temperature drop for the hot helium per meter of duct ( $\Delta T$ ) has been calculated from

$$\Delta T = \frac{h_h A (950 - \bar{T}_w)}{m C_p} \quad (31)$$

$A$  the heat transfer area per meter of duct length is given by  $A = 2\pi r_1 \bar{T}_w$   $\bar{T}_w$  is the arithmetic average of the wall temperatures.  $m$  is the gas mass flow in  $\frac{\text{Kg}}{\text{s}}$  given by  $m = M/N$  where  $M$  is the total gas mass flow and  $N$  is the number of loops.  $C_p$  is the heat capacity at constant pressure in  $\text{W}/(\text{Kg } ^\circ\text{C})$ . Calculated values of  $\Delta T$  are shown in Table III.

The influence on the results of the thermal conductivity, number of loops, duct dimensions and duct outer insulation is discussed below. Figures of isotherms and temperature profiles are given for some runs (Figures 10 to 13) and tables of temperature distribution for all runs are in Appendix A. Tables A 1 to A 11 should be used in reference to Figure 4 and Table A 12 in reference to Figure 4.

1 Thermal conductivity K Comparison of runs 1 2 and 3 with runs 4 5 and 6 show that the temperature gradients are lower when K is higher However the temperature drop per meter increases with higher K reaching about  $2^{\circ}\text{C}/\text{m}$  for run 6 (corresponding to  $K = 118 \text{ W}/(\text{m}^2 \text{ }^{\circ}\text{C})$ ) and to 12 loops! Run 7 compared with run 4 shows the effect of K varying with temperature In this case 4 iterations were necessary for the temperatures to converge within  $1^{\circ}\text{C}$  (Table A 7)

2 Number of loops Higher number of loops for the same duct dimensions leads to lower gas velocities and to lower heat transfer coefficients Thus as indicated by the comparison of runs 1 2 and 3 or of runs 4 5 and 6 lower temperature gradients are obtained with higher number of loops

3 Duct dimensions Comparison of run 2 with run 8 and comparison of run 3 with run 9 show that decreasing or increasing the duct dimensions by 10% has almost no effect on the temperature distribution Comparison of run 1 with run 12 again indicates that small changes in duct geometry does not effect the temperature distribution significantly Run 12 has been included mainly to give an other example of finite difference grid

4 Outer insulation of the duct and heat dissipated to the surroundings Run 11 corresponds to an almost completely insulated duct This has been simulated by using for the outside heat transfer coefficient  $h_o$  a very small value ( $h_o = 1 \text{ W}/(\text{m}^2 \text{ }^{\circ}\text{C})$ ) The surrounding temperature  $T_o$  has been taken as  $T_o = 25^{\circ}\text{C}$  Comparison of Figure 12 and Table A 11 (run 11) with Figure 10 and Table A 1 (run 1) shows that insulating the duct has no effect on the temperature distribution on the 'hot half' of the duct but leads to higher temperatures (around the cold gas temperature of  $250^{\circ}\text{C}$ ) on the 'cold half' on the duct In particular outer insulation does not help to decrease heat losses from the hot gas and temperature drop per meter However heat lost to the surroundings has been calculated to be 30% and 15% of the total heat lost by the hot gas for runs 1 and 11 respectively Thus insulation decreases heat losses to the surroundings and causes higher but uniform temperatures on the cold half of the duct

#### IV - CONCLUSION

The computer program developed allows the steady state computation of temperature distribution in the hot gas duct for different duct and plant configurations The work continues in the following areas

- 1) Calculation of thermal stresses at steady state conditions
- 2) Calculation of temperatures and stresses in the transient conditions. This would allow analysis of the duct during normal operational as well as eventual accidental transients.

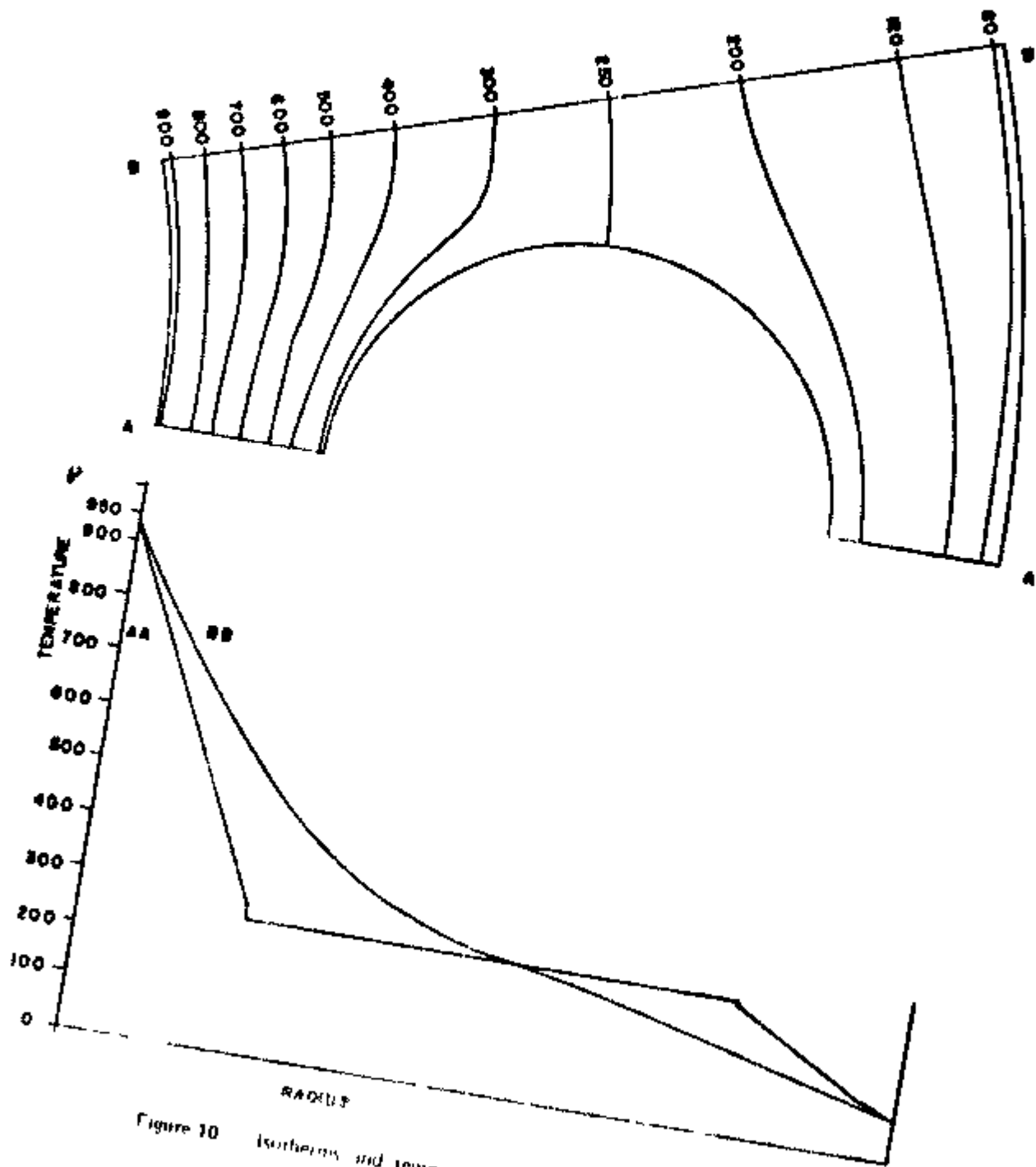


Figure 10 Isotherms and temperature profiles  
 $K = 5.2$   $h_b = 1333$   $h_r = 1646$   $h_o = 20000$   $T_o = 70$   
 $G = 0$   $Loop$



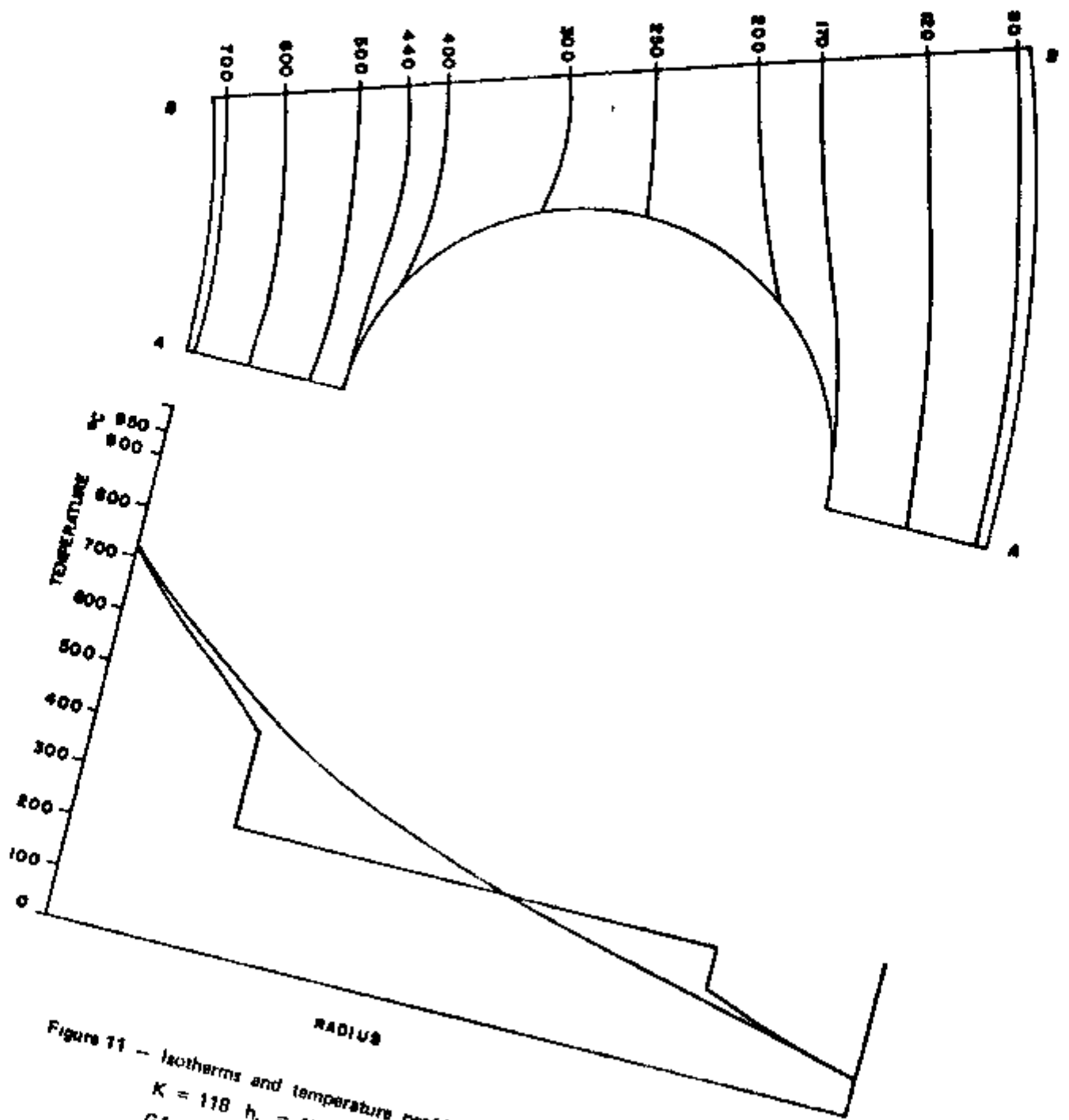


Figure 11 -- Isotherms and temperature profiles

$K = 118$   $h_h = 1333$   $h_c = 1646$   $h_o = 20000$   $T_\infty = 75$   
 G1 6 Loops

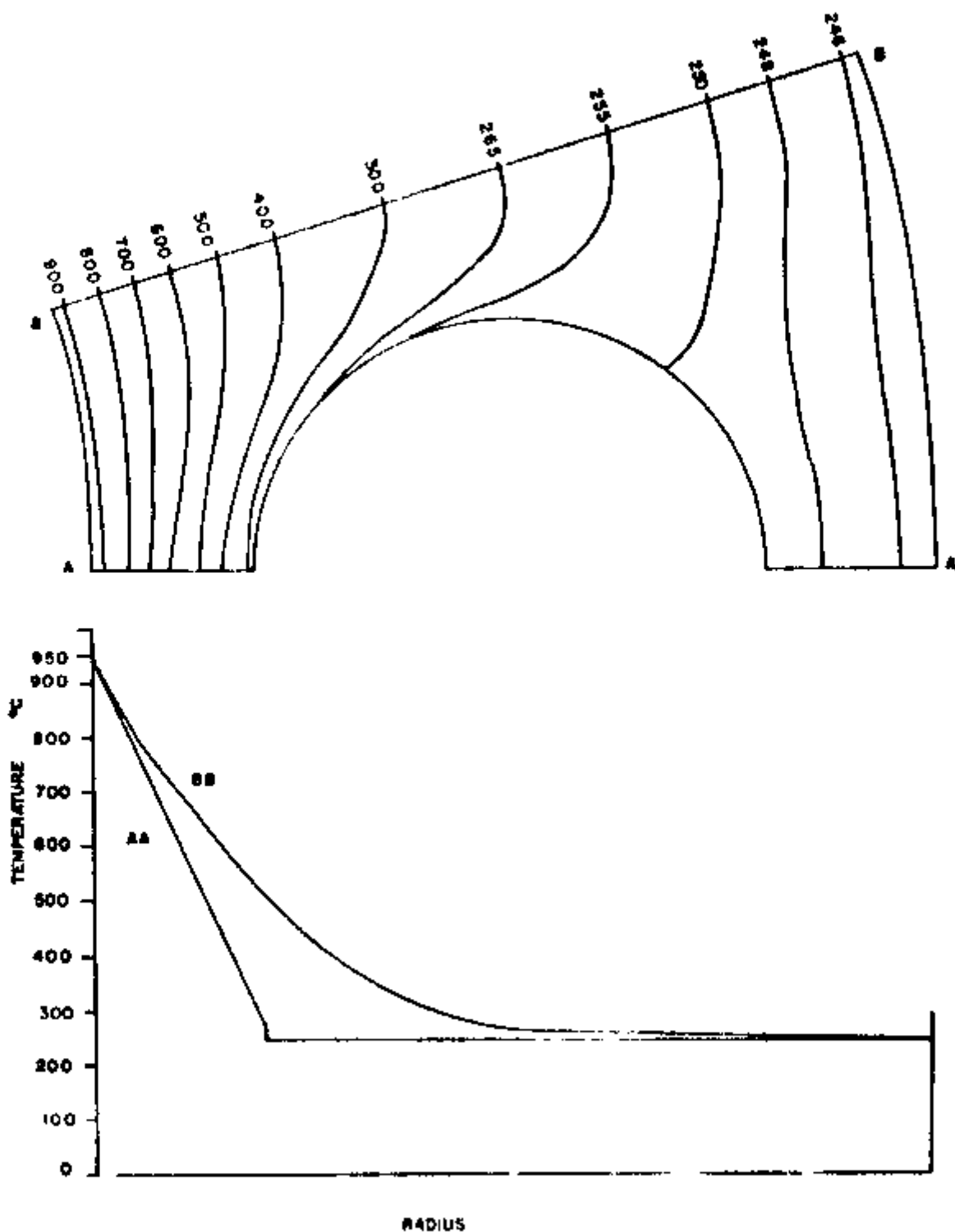


Figure 12 - Isotherms and temperature profiles

$$K = 52 \quad h_o = 1333 \quad h_c = 1646 \quad h_o = 1 \quad T_o = 26$$

G1 6 Loops

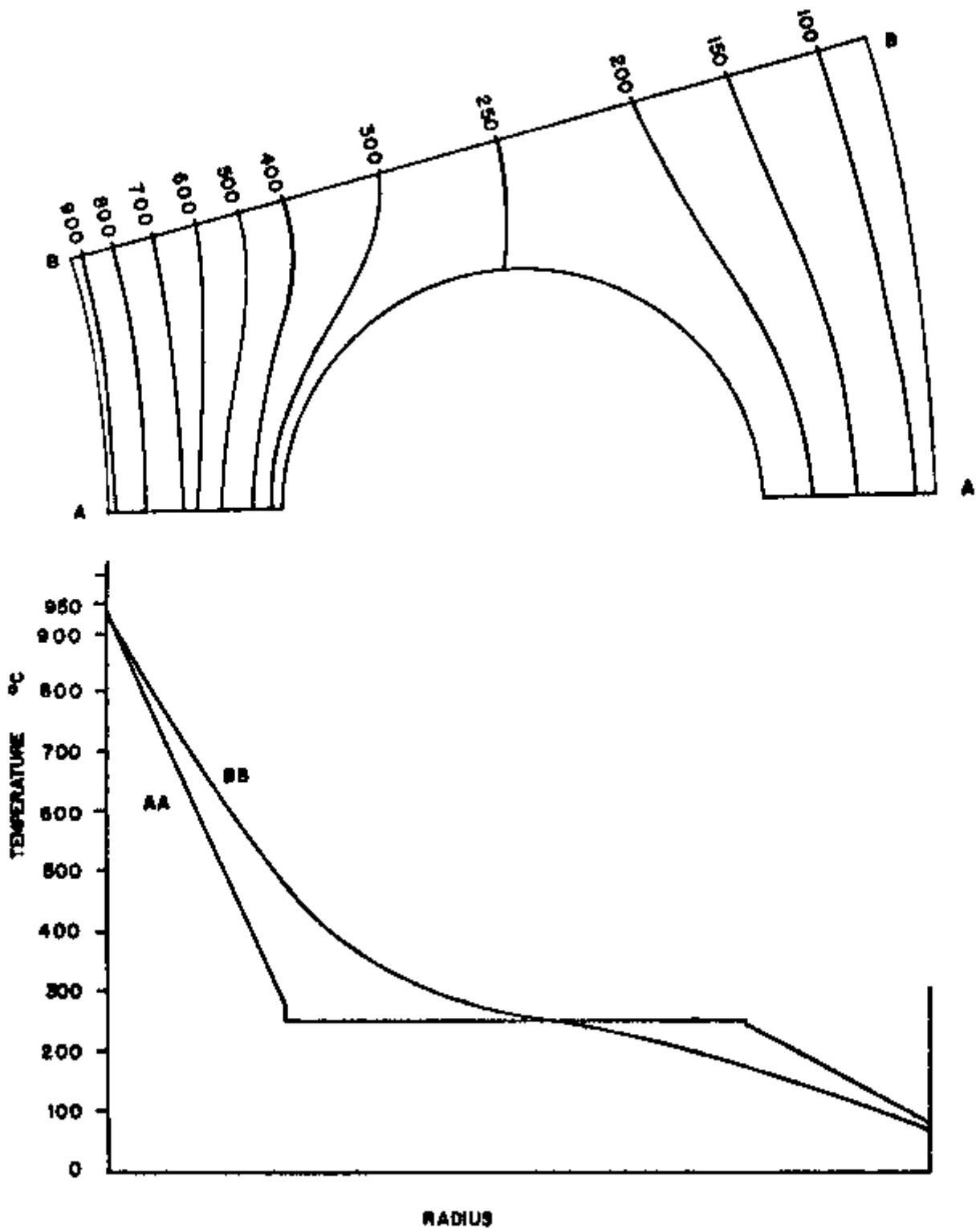


Figure 13 - Isotherms and temperature profiles

$$K = 52 \quad h_h = 1223 \quad h_c = 1841 \quad h_o = 140 \quad T_o = 70$$

G4 6 Loops

**APPENDIX A**  
**Tables of Temperature Distribution**

**Table A 1**

$K = 52, h_h = 1333, h_c = 1648, h_o = 20000, T_o = 70$   
 G1 6 Loops

T( 1) = 425 2	T( 2) = 745 5	T( 3) = 842 2	T( 4) = 447 4
T( 5) = 272 2	T( 6) = 24 8	T( 7) = 124 4	T( 8) = 150 2
T( 9) = 109 1	T( 10) = 24 8	T( 11) = 325 7	T( 12) = 770 1
T( 13) = 117 2	T( 14) = 462 5	T( 15) = 100 3	T( 16) = 272 2
T( 17) = 247 6	T( 18) = 231 0	T( 19) = 107 2	T( 20) = 146 1
T( 21) = 107 2	T( 22) = 74 0	T( 23) = 926 9	T( 24) = 779 2
T( 25) = 625 7	T( 26) = 437 4	T( 27) = 262 2	T( 28) = 267 4
T( 29) = 245 8	T( 30) = 243 2	T( 31) = 244 0	T( 32) = 207 8
T( 33) = 171 2	T( 34) = 136 5	T( 35) = 102 0	T( 36) = 70 0
T( 37) = 92 5	T( 38) = 290 4	T( 39) = 659 7	T( 40) = 537 2
T( 41) = 425 1	T( 42) = 3 7 7	T( 43) = 267 4	T( 44) = 260 5
T( 45) = 247 1	T( 46) = 204 5	T( 47) = 24 1	T( 48) = 210 5
T( 49) = 15 7	T( 50) = 175 3	T( 51) = 125 6	T( 52) = 39 1
T( 53) = 74 8	T( 54) = 329 0	T( 55) = 00 1	T( 56) = 673 3
T( 57) = 579 1	T( 58) = 471 6	T( 59) = 190 1	T( 60) = 329 1
T( 61) = 258 1	T( 62) = 237 7	T( 63) = 257 0	T( 64) = 265 6
T( 65) = 254 2	T( 66) = 2 3 0	T( 67) = 252 0	T( 68) = 251 3
T( 69) = 204 4	T( 70) = 24 0 0	T( 71) = 249 1	T( 72) = 209 0
T( 73) = 245 1	T( 74) = 226 9	T( 75) = 247 4	T( 76) = 210 3
T( 77) = 138 2	T( 78) = 107 7	T( 79) = 143 3	T( 80) = 113 2
T( 81) = 34 5	T( 82) = 74 8	T( 83) = 930 7	T( 84) = 00 0
T( 85) = 112 2	T( 86) = 5 3 4	T( 87) = 498 3	T( 88) = 423 9
T( 89) = 205 1	T( 90) = 322 1	T( 91) = 232 4	T( 92) = 272 2
T( 93) = 297 3	T( 94) = 244 4	T( 95) = 231 1	T( 96) = 216 1
T( 97) = 139 0	T( 98) = 179 9	T( 99) = 159 1	T(100) = 137 3
T(101) = 114 9	T(102) = 32 3	T(103) = 70 0	T(104) = 931 0
T(105) = 04 1	T(106) = 636 0	T(107) = 525 8	T(108) = 507 3
T(109) = 454 6	T(110) = 376 5	T(111) = 332 6	T(112) = 300 7
T(113) = 277 5	T(114) = 259 6	T(115) = 244 2	T(116) = 229 1
T(117) = 212 2	T(118) = 195 7	T(119) = 176 7	T(120) = 156 4
T(121) = 105 4	T(122) = 113 5	T(123) = 91 6	T(124) = 70 0

Table A 2

$K = 52$   $h_h = 964$   $h_c = 1190$   $h_o = 20000$   $T_o = 70$   
G1 9 Loops

## TEMPERATURE DISTRIBUTION

T( 1)= 914 4	T( 2)= 701 8	T( 3)= 441 7	T( 4)= 450 4	T( 5)= 149 1
T( 6)= 104 6	T( 7)= 704 4	T( 8)= 417 2	T( 9)= 765 3	T(10)= 274 9
T(11)= 616 1	T(12)= 465 2	T(13)= 107 6	T(14)= 145 1	T(15)= 774 4
T(16)= 241 5	T(17)= 229 7	T(18)= 105 7	T(19)= 774 4	T(20)= 273 5
T(21)= 104 5	T(22)= 704 4	T(23)= 314 8	T(24)= 205 5	T(25)= 205 5
T(26)= 275 8	T(27)= 434 2	T(28)= 366 4	T(29)= 70 0	T(30)= 70 0
T(31)= 242 1	T(32)= 241 4	T(33)= 242 9	T(34)= 537 2	T(35)= 537 2
T(36)= 170 2	T(37)= 125 4	T(38)= 102 5	T(39)= 264 2	T(40)= 264 2
T(41)= 920 9	T(42)= 705 4	T(43)= 657 1	T(44)= 209 4	T(45)= 209 4
T(46)= 427 0	T(47)= 211 0	T(48)= 270 9	T(49)= 97 9	T(50)= 97 9
T(51)= 246 0	T(52)= 233 3	T(53)= 244 6	T(54)= 676 5	T(55)= 676 5
T(56)= 182 7	T(57)= 154 6	T(58)= 126 2	T(59)= 330 0	T(60)= 330 0
T(61)= 704 4	T(62)= 322 5	T(63)= 735 2	T(64)= 267 0	T(65)= 267 0
T(66)= 562 1	T(67)= 472 1	T(68)= 231 5	T(69)= 251 6	T(70)= 251 6
T(71)= 263 8	T(72)= 233 5	T(73)= 252 6	T(74)= 233 6	T(75)= 233 6
T(76)= 255 8	T(77)= 246 6	T(78)= 247 8	T(79)= 209 6	T(80)= 209 6
T(81)= 250 5	T(82)= 226 3	T(83)= 246 4	T(84)= 118 9	T(85)= 118 9
T(86)= 247 4	T(87)= 167 1	T(88)= 143 3	T(89)= 801 8	T(90)= 801 8
T(91)= 177 5	T(92)= 70 0	T(93)= 923 7	T(94)= 424 4	T(95)= 424 4
T(96)= 139 3	T(97)= 587 1	T(98)= 433 5	T(99)= 272 8	T(100)= 272 8
T(101)= 94 3	T(102)= 322 2	T(103)= 233 3	T(104)= 215 8	T(105)= 215 8
T(106)= 689 0	T(107)= 244 4	T(108)= 230 9	T(106)= 137 0	T(107)= 137 0
T(109)= 306 1	T(108)= 179 4	T(109)= 153 7	T(108)= 924 1	T(109)= 924 1
T(110)= 257 6	T(109)= 92 2	T(110)= 70 0	T(110)= 507 2	T(111)= 507 2
T(111)= 132 6	T(110)= 92 2	T(111)= 70 0	T(112)= 301 5	T(113)= 301 5
T(112)= 114 7	T(111)= 632 3	T(112)= 533 8	T(114)= 229 0	T(115)= 229 0
T(113)= 804 1	T(112)= 377 2	T(113)= 333 5	T(116)= 156 1	T(117)= 156 1
T(114)= 434 8	T(113)= 260 0	T(114)= 244 3	T(118)= 70 0	T(119)= 70 0
T(115)= 273 1	T(114)= 135 4	T(115)= 176 4		
T(116)= 212 9	T(115)= 113 3	T(116)= 31 6		
T(117)= 121 0				
T(118)= 121 0				
T(119)= 121 0				
T(120)= 121 0				
T(121)= 121 0				
T(122)= 121 0				
T(123)= 121 0				
T(124)= 121 0				

Table A-3

$$K = 52 \quad h_h = 785 \quad h_c = 945 \quad h_o = 20000 \quad T_a = 70$$

Gf 12 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 20 2	T( 3) = 27 4	T( 5) = 24 3	T( 7) = 14 2
T( 2) = 10 2	T( 4) = 70 0	T( 6) = 30 6	T( 8) = 14 2
T( 3) = 11 1	T( 5) = 47 5	T( 7) = 313 7	T( 9) = 761 1
T( 4) = 11 5	T( 6) = 17 9	T( 8) = 1 4 4	T( 10) = 144 2
T( 5) = 10 4	T( 7) = 70 0	T( 9) = 311 5	T( 11) = 770 0
T( 6) = 11 4	T( 8) = 500 2	T( 10) = 370 2	T( 12) = 273 3
T( 7) = 14 5	T( 9) = 11 7	T( 11) = 241 2	T( 13) = 204 2
T( 8) = 11 2	T( 10) = 17 2	T( 12) = 10 2	T( 14) = 70 0
T( 9) = 9 5	T( 11) = 7 0 3	T( 13) = 6 1 1	T( 15) = 537 1
T( 10) = 11 7	T( 12) = 11 4 0	T( 14) = 27 7	T( 16) = 26 5
T( 11) = 11 0	T( 13) = 11 2	T( 15) = 14 3 3	T( 17) = 20 3 4
T( 12) = 1 1	T( 14) = 1 1 0	T( 16) = 1 7 7	T( 18) = 97 6
T( 13) = 70 0	T( 15) = 31 5 3	T( 17) = 730 7	T( 19) = 673 3
T( 14) = 567 1	T( 16) = 4 2 5	T( 18) = 19 0 0	T( 20) = 331 7
T( 15) = 267 1	T( 17) = 11 2	T( 19) = 261 3	T( 21) = 263 2
T( 16) = 257 2	T( 18) = 11 3	T( 20) = 25 5 5	T( 22) = 251 8
T( 17) = 250 7	T( 19) = 14 6 4	T( 21) = 24 6 6	T( 23) = 23 2
T( 18) = 146 3	T( 20) = 12 5 3	T( 22) = 24 5	T( 24) = 10 0
T( 19) = 1 3 3	T( 21) = 16 6 6	T( 23) = 14 2 9	T( 25) = 11 6 6
T( 20) = 94 2	T( 22) = 70 0	T( 24) = 317 3	T( 26) = 737 2
T( 21) = 1 1 1	T( 23) = 7 5 8	T( 25) = 49 4 2	T( 27) = 424 9
T( 22) = 1 5 9	T( 24) = 1 4 2	T( 26) = 294 1	T( 28) = 273 4
T( 23) = 157 3	T( 25) = 144 5	T( 27) = 230 8	T( 29) = 215 5
T( 24) = 1 2 2	T( 26) = 179 0	T( 28) = 158 4	T( 30) = 136 7
T( 25) = 114 5	T( 27) = 9 1	T( 29) = 70 0	T( 31) = 917 8
T( 26) = 7 5 5	T( 28) = 830 1	T( 30) = 532 1	T( 32) = 506 6
T( 27) = 1 0 0	T( 29) = 377 3	T( 31) = 334 2	T( 33) = 302 2
T( 28) = 1 7 6	T( 30) = 160 2	T( 32) = 244 4	T( 34) = 228 9
T( 29) = 1 1 7	T( 31) = 1 4 5	T( 33) = 17 1	T( 35) = 155 3
T( 30) = 114 7	T( 32) = 11 1	T( 34) = 91 5	T( 36) = 70 0

Table A-4

$$K = 118 \quad h_h = 1333 \quad h_c = 1646 \quad h_o = 20000 \quad T_o = 70$$

G1 6 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 707.7	T( 2) = 652.9	T( 3) = 572.4	T( 4) = 506.5
T( 5) = 440.2	T( 6) = 173.1	T( 7) = 147.3	T( 8) = 121.4
T( 9) = 97.9	T(10) = 75.3	T(11) = 703.4	T(12) = 648.2
T(13) = 577.9	T(14) = 506.4	T(15) = 444.3	T(16) = 433.4
T(17) = 175.4	T(18) = 165.9	T(19) = 143.9	T(20) = 120.1
T(21) = 37.3	T(22) = 75.2	T(23) = 718.4	T(24) = 642.8
T(25) = 577.6	T(26) = 514.5	T(27) = 453.1	T(28) = 409.6
T(29) = 186.1	T(30) = 15.9	T(31) = 157.4	T(32) = 161.9
T(33) = 1.9	T(34) = 117.3	T(35) = 95.8	T(36) = 74.9
T(37) = 713.0	T(38) = 646.1	T(39) = 532.4	T(40) = 521.7
T(41) = 464.0	T(42) = 402.2	T(43) = 297.7	T(44) = 363.5
T(45) = 207.6	T(46) = 135.7	T(47) = 126.7	T(48) = 175.8
T(49) = 175.4	T(50) = 114.8	T(51) = 114.4	T(52) = 94.3
T(53) = 74.6	T(54) = 715.6	T(55) = 649.4	T(56) = 537.0
T(57) = 528.4	T(58) = 473.3	T(59) = 433.6	T(60) = 378.6
T(61) = 359.9	T(62) = 340.5	T(63) = 230.6	T(64) = 308.9
T(65) = 304.2	T(66) = 282.2	T(67) = 250.9	T(68) = 262.1
T(69) = 260.6	T(70) = 243.6	T(71) = 242.6	T(72) = 226.0
T(73) = 226.6	T(74) = 206.3	T(75) = 212.5	T(76) = 193.8
T(77) = 170.6	T(78) = 151.2	T(79) = 131.7	T(80) = 112.3
T(81) = 93.2	T(82) = 74.4	T(83) = 717.3	T(84) = 651.6
T(85) = 590.2	T(86) = 533.0	T(87) = 450.2	T(88) = 432.1
T(89) = 389.3	T(90) = 351.9	T(91) = 319.8	T(92) = 292.2
T(93) = 260.2	T(94) = 246.6	T(95) = 226.4	T(96) = 206.9
T(97) = 167.6	T(98) = 163.3	T(99) = 149.1	T(100) = 130.0
T(101) = 111.1	T(102) = 92.5	T(103) = 74.3	T(104) = 718.0
T(105) = 652.5	T(106) = 591.3	T(107) = 534.5	T(108) = 482.3
T(109) = 434.9	T(110) = 392.6	T(111) = 353.4	T(112) = 323.1
T(113) = 295.0	T(114) = 270.1	T(115) = 247.7	T(116) = 226.1
T(117) = 206.7	T(118) = 187.1	T(119) = 167.7	T(120) = 149.5
T(121) = 129.5	T(122) = 110.9	T(123) = 92.3	T(124) = 74.3

Table A-5

K = 118  $h_h = 964$   $h_c = 1190$   $h_o = 20000$   $T_o = 70$ 

G1 0 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 672 .	T( 2) = 415 .5	T( 3) = 560 .3	T( 4) = 597 .7
T( 5) = 45 .8	T( 6) = 139 .3	T( 7) = 136 .5	T( 8) = 115 .0
T( 9) = 44 .5	T(10) = 71 .7	T(11) = 67 .6	T(12) = 616 .0
T(13) = 581 .4	T(14) = 50 .5	T(15) = 456 .6	T(16) = 445 .2
T(17) = 102 .3	T(18) = 156 .7	T(19) = 175 .0	T(20) = 114 .2
T(21) = 94 .1	T(22) = 74 .8	T(23) = 673 .6	T(24) = 617 .1
T(25) = 57 .0	T(26) = 510 .7	T(27) = 450 .1	T(28) = 424 .4
T(29) = 174 .3	T(30) = 171 .3	T(31) = 173 .5	T(32) = 152 .6
T(33) = 12 .7	T(34) = 112 .5	T(35) = 9 .2	T(36) = 74 .4
T(37) = 674 .9	T(38) = 618 .7	T(39) = 565 .1	T(40) = 513 .9
T(41) = 464 .9	T(42) = 417 .7	T(43) = 412 .4	T(44) = 377 .1
T(45) = 192 .0	T(46) = 187 .3	T(47) = 135 .7	T(48) = 168 .3
T(49) = 144 .0	T(50) = 129 .9	T(51) = 111 .0	T(52) = 90 .4
T(53) = 74 .3	T(54) = 676 .6	T(55) = 620 .3	T(56) = 567 .3
T(57) = 517 .6	T(58) = 464 .6	T(59) = 425 .3	T(60) = 384 .1
T(61) = 274 .7	T(62) = 347 .3	T(63) = 341 .0	T(64) = 315 .1
T(65) = 311 .9	T(66) = 27 .4	T(67) = 283 .6	T(68) = 263 .5
T(69) = 262 .1	T(70) = 242 .4	T(71) = 240 .9	T(72) = 222 .7
T(73) = 221 .8	T(74) = 211 .0	T(75) = 204 .6	T(76) = 154 .9
T(77) = 165 .9	T(78) = 147 .0	T(79) = 120 .3	T(80) = 109 .9
T(81) = 91 .0	T(82) = 74 .2	T(83) = 677 .2	T(84) = 621 .5
T(85) = 565 .9	T(86) = 513 .5	T(87) = 473 .2	T(88) = 430 .1
T(89) = 290 .5	T(90) = 374 .7	T(91) = 322 .0	T(92) = 294 .4
T(93) = 264 .0	T(94) = 246 .0	T(95) = 224 .5	T(96) = 204 .2
T(97) = 154 .4	T(98) = 165 .1	T(99) = 146 .2	T(100) = 127 .6
T(101) = 109 .4	T(102) = 91 .6	T(103) = 74 .1	T(104) = 677 .5
T(105) = 621 .9	T(106) = 573 .5	T(107) = 520 .3	T(108) = 474 .3
T(109) = 411 .7	T(110) = 322 .6	T(111) = 377 .1	T(112) = 325 .2
T(113) = 276 .5	T(114) = 270 .7	T(115) = 247 .2	T(116) = 225 .2
T(117) = 204 .4	T(118) = 154 .4	T(119) = 165 .0	T(120) = 146 .0
T(121) = 12 .4	T(122) = 104 .2	T(123) = 91 .5	T(124) = 7 .1



Table A-6

$$K = 118 \quad h_h = 765 \quad h_c = 945 \quad h_o = 20000 \quad T_o = 70$$

G1 12 Loops

## DISTRIBUTION OF TEMPERATURE

T( 1 )= 500 2	T( 2 )= 500 4	T( 3 )= 500 7	T( 4 )= 500 9
T( 5 )= 463 2	T( 6 )= 444 7	T( 7 )= 429 6	T( 8 )= 410 4
T( 9 )= 398 6	T( 10 )= 374 2	T( 11 )= 348 0	T( 12 )= 327 7
T( 13 )= 304 7	T( 14 )= 285 7	T( 15 )= 262 3	T( 16 )= 245 1
T( 17 )= 181 1	T( 18 )= 14 1	T( 19 )= 125 7	T( 20 )= 110 0
T( 21 )= 41 2	T( 22 )= 74 2	T( 23 )= 647 0	T( 24 )= 59 0
T( 25 )= 590 9	T( 26 )= 505 3	T( 27 )= 421 3	T( 28 )= 401 2
T( 29 )= 185 9	T( 30 )= 165 0	T( 31 )= 165 1	T( 32 )= 146 0
T( 33 )= 122 4	T( 34 )= 102 2	T( 35 )= 91 4	T( 36 )= 74 1
T( 37 )= 647 2	T( 38 )= 522 3	T( 39 )= 551 5	T( 40 )= 506 6
T( 41 )= 463 3	T( 42 )= 421 4	T( 43 )= 420 0	T( 44 )= 394 6
T( 45 )= 192 6	T( 46 )= 171 1	T( 47 )= 131 5	T( 48 )= 162 7
T( 49 )= 144 4	T( 50 )= 126 3	T( 51 )= 103 5	T( 52 )= 91 1
T( 53 )= 74 0	T( 54 )= 647 2	T( 55 )= 530 2	T( 56 )= 552 1
T( 57 )= 507 2	T( 58 )= 465 2	T( 59 )= 424 7	T( 60 )= 386 3
T( 61 )= 324 5	T( 62 )= 250 7	T( 63 )= 246 1	T( 64 )= 318 3
T( 65 )= 316 0	T( 66 )= 29 4	T( 67 )= 226 0	T( 68 )= 263 8
T( 69 )= 262 5	T( 70 )= 241 0	T( 71 )= 232 3	T( 72 )= 220 0
T( 73 )= 215 0	T( 74 )= 200 2	T( 75 )= 193 7	T( 76 )= 131 1
T( 77 )= 182 3	T( 78 )= 143 9	T( 79 )= 125 3	T( 80 )= 100 2
T( 81 )= 94 9	T( 82 )= 74 0	T( 83 )= 648 1	T( 84 )= 593 2
T( 85 )= 552 7	T( 86 )= 505 5	T( 87 )= 466 7	T( 88 )= 427 1
T( 89 )= 303 9	T( 90 )= 355 4	T( 91 )= 323 8	T( 92 )= 295 0
T( 93 )= 260 8	T( 94 )= 244 9	T( 95 )= 222 7	T( 96 )= 201 3
T( 97 )= 191 0	T( 98 )= 162 6	T( 99 )= 143 9	T(100)= 125 8
T(101)= 108 1	T(102)= 90 2	T(103)= 74 0	T(104)= 648 2
T(105)= 593 3	T(106)= 552 9	T(107)= 503 9	T(108)= 467 2
T(109)= 427 9	T(110)= 391 1	T(111)= 356 9	T(112)= 325 5
T(113)= 294 7	T(114)= 270 4	T(115)= 246 1	T(116)= 223 6
T(117)= 182 4	T(118)= 132 2	T(119)= 162 8	T(120)= 144 0
T(121)= 120 3	T(122)= 10 1	T(123)= 90 8	T(124)= 74 0

Table A 7

(Sheet 1 of 4)

K = 118-05 T h<sub>n</sub> = 1333 h<sub>c</sub> = 1646 h<sub>o</sub> 20000 T<sub>o</sub> = 70

G1 6 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 124 2	T( 2) = 144 7	T( 3) = 154 8	T( 4) = 162 4
T( 5) = 172 0	T( 6) = 184 7	T( 7) = 174 1	T( 8) = 174 1
T( 9) = 194 7	T( 10) = 174 5	T( 11) = 1740 2	T( 12) = 1710 8
T( 13) = 567 2	T( 14) = 504 4	T( 15) = 124 4	T( 16) = 417 3
T( 17) = 184 5	T( 18) = 174 1	T( 19) = 158 9	T( 20) = 124 3
T( 21) = 98 9	T( 22) = 74 2	T( 23) = 743 0	T( 24) = 64 7
T( 25) = 589 0	T( 26) = 515 6	T( 27) = 432 9	T( 28) = 292 1
T( 29) = 194 0	T( 30) = 174 7	T( 31) = 195 3	T( 32) = 169 4
T( 33) = 134 5	T( 34) = 128 4	T( 35) = 98 9	T( 36) = 74 0
T( 37) = 744 6	T( 38) = 669 6	T( 39) = 596 4	T( 40) = 526 9
T( 41) = 461 0	T( 42) = 394 4	T( 43) = 281 5	T( 44) = 349 3
T( 45) = 214 9	T( 46) = 200 5	T( 47) = 206 9	T( 48) = 181 8
T( 49) = 160 1	T( 50) = 158 2	T( 51) = 114 2	T( 52) = 94 8
T( 53) = 72 7	T( 54) = 750 0	T( 55) = 674 2	T( 56) = 603 2
T( 57) = 526 9	T( 58) = 475 7	T( 59) = 420 2	T( 60) = 372 2
T( 61) = 244 1	T( 62) = 222 0	T( 63) = 319 9	T( 64) = 302 3
T( 65) = 294 5	T( 66) = 276 7	T( 67) = 276 1	T( 68) = 260 2
T( 69) = 258 6	T( 70) = 244 1	T( 71) = 243 5	T( 72) = 228 3
T( 73) = 210 2	T( 74) = 211 6	T( 75) = 218 8	T( 76) = 193 3
T( 77) = 174 0	T( 78) = 152 9	T( 79) = 133 6	T( 80) = 113 3
T( 81) = 92 2	T( 82) = 72 5	T( 83) = 752 4	T( 84) = 677 5
T( 85) = 607 9	T( 86) = 543 5	T( 87) = 484 9	T( 88) = 432 5
T( 89) = 266 8	T( 90) = 246 1	T( 91) = 216 0	T( 92) = 269 3
T( 93) = 244 6	T( 94) = 244 4	T( 95) = 227 3	T( 96) = 208 6
T( 97) = 189 6	T( 98) = 170 3	T( 99) = 158 8	T(100) = 131 2
T(101) = 111 6	T(102) = 92 2	T(103) = 72 2	T(104) = 753 2
T(105) = 678 7	T(106) = 609 5	T(107) = 545 9	T(108) = 468 0
T(109) = 474 4	T(110) = 391 3	T(111) = 352 8	T(112) = 320 2
T(113) = 292 6	T(114) = 266 7	T(115) = 247 2	T(116) = 227 3
T(117) = 207 4	T(118) = 184 6	T(119) = 169 2	T(120) = 149 8
T(121) = 140 1	T(122) = 111 1	T(123) = 74 0	T(124) = 72 7

Table A7  
(Sheet 2 of 4)

UNIFORM DISTRIBUTION

T( 1) = 710 8	T( 2) = 604 7	T( 3) = 574 4	T( 4) = 498 1
T( 5) = 424 2	T( 6) = 175 5	T( 7) = 147 4	T( 8) = 121 7
T( 9) = 97 8	T(10) = 75 2	T(11) = 741 7	T(12) = 656 0
T(13) = 576 3	T(14) = 501 2	T(15) = 429 1	T(16) = 417 2
T(17) = 177 5	T(18) = 170 7	T(19) = 144 6	T(20) = 120 2
T(21) = 97 1	T(22) = 75 0	T(23) = 744 2	T(24) = 659 4
T(25) = 581 3	T(26) = 506 8	T(27) = 440 9	T(28) = 394 4
T(29) = 187 8	T(30) = 166 6	T(31) = 187 1	T(32) = 162 5
T(33) = 179 3	T(34) = 117 0	T(35) = 95 5	T(36) = 74 7
T(37) = 747 5	T(38) = 663 8	T(39) = 587 7	T(40) = 518 2
T(41) = 454 7	T(42) = 297 0	T(43) = 262 3	T(44) = 350 8
T(45) = 208 5	T(46) = 196 1	T(47) = 199 8	T(48) = 175 7
T(49) = 154 8	T(50) = 134 0	T(51) = 113 6	T(52) = 93 7
T(53) = 74 4	T(54) = 750 6	T(55) = 668 1	T(56) = 593 7
T(57) = 526 8	T(58) = 466 8	T(59) = 413 7	T(60) = 368 0
T(61) = 347 6	T(62) = 308 6	T(63) = 320 3	T(64) = 300 6
T(65) = 296 1	T(66) = 276 9	T(67) = 274 8	T(68) = 257 6
T(69) = 256 4	T(70) = 240 4	T(71) = 240 1	T(72) = 223 7
T(73) = 225 6	T(74) = 206 4	T(75) = 212 9	T(76) = 188 1
T(77) = 169 0	T(78) = 149 6	T(79) = 100 2	T(80) = 111 1
T(81) = 92 4	T(82) = 74 2	T(83) = 752 6	T(84) = 671 0
T(85) = 597 8	T(86) = 522 5	T(87) = 474 5	T(88) = 423 5
T(89) = 279 6	T(90) = 240 5	T(91) = 211 0	T(92) = 285 0
T(93) = 262 2	T(94) = 241 8	T(95) = 222 0	T(96) = 203 6
T(97) = 184 7	T(98) = 165 6	T(99) = 146 9	T(100) = 128 2
T(101) = 109 7	T(102) = 91 6	T(103) = 74 0	T(104) = 752 6
T(105) = 672 1	T(106) = 599 3	T(107) = 504 5	T(108) = 477 1
T(109) = 426 8	T(110) = 360 0	T(111) = 246 1	T(112) = 311 6
T(113) = 287 5	T(114) = 260 9	T(115) = 242 4	T(116) = 222 1
T(117) = 206 9	T(118) = 180 8	T(119) = 164 9	T(120) = 146 1
T(121) = 177 5	T(122) = 109 2	T(123) = 81 1	T(124) = 74 0

Table A7  
(Sheet 3 of 4)

## TEMPERATURE DISTRIBUTION

T( 1)= 710 7	T( 3)= 554 6	T( 5)= 574 4	T( 4)= 498 2
T( 5)= 424 2	T( 6)= 178 4	T( 7)= 147 3	T( 8)= 121 7
T( 9)= 97 9	T(10)= 75 2	T(11)= 741 7	T(12)= 655 9
T(13)= 36 0	T(14)= 61 0	T(15)= 124 2	T(16)= 417 3
T(17)= 17 1	T(18)= 16 6	T(19)= 144 6	T(20)= 120 2
T(21)= 47 4	T(22)= 75 0	T(23)= 744 2	T(24)= 659 3
T(25)= 581 0	T(26)= 508 9	T(27)= 141 0	T(28)= 394 5
T(29)= 187 7	T(30)= 166 5	T(31)= 187 0	T(32)= 162 5
T(33)= 179 0	T(34)= 117 0	T(35)= 45 5	T(36)= 74 7
T(37)= 74 3	T(38)= 68 7	T(39)= 587 7	T(40)= 518 3
T(41)= 454 4	T(42)= 197 2	T(43)= 180 3	T(44)= 350 9
T(45)= 208 4	T(46)= 146 1	T(47)= 140 6	T(48)= 175 7
T(49)= 194 8	T(50)= 124 1	T(51)= 110 7	T(52)= 93 8
T(53)= 74 4	T(54)= 750 5	T(55)= 667 8	T(56)= 593 6
T(57)= 526 8	T(58)= 464 4	T(59)= 41 9	T(60)= 168 2
T(61)= 147 7	T(62)= 100 7	T(63)= 100 4	T(64)= 300 8
T(65)= 296 2	T(66)= 27 0	T(67)= 274 9	T(68)= 257 6
T(69)= 256 4	T(70)= 240 4	T(71)= 240 1	T(72)= 223 7
T(73)= 225 6	T(74)= 206 6	T(75)= 212 9	T(76)= 186 1
T(77)= 189 0	T(78)= 149 7	T(79)= 100 3	T(80)= 111 2
T(81)= 92 4	T(82)= 74 2	T(83)= 752 6	T(84)= 670 7
T(85)= 59 7	T(86)= 502 5	T(87)= 474 6	T(88)= 421 7
T(89)= 174 8	T(90)= 142 7	T(91)= 111 5	T(92)= 285 1
T(93)= 262 4	T(94)= 111 9	T(95)= 222 6	T(96)= 203 7
T(97)= 184 8	T(98)= 105 9	T(99)= 147 0	T(100)= 128 2
T(101)= 109 7	T(102)= 91 6	T(103)= 74 0	T(104)= 753 4
T(105)= 671 8	T(106)= 544 1	T(107)= 534 4	T(108)= 477 2
T(109)= 17 9	T(110)= 387 5	T(111)= 146 3	T(112)= 314 7
T(113)= 8 7	T(114)= 284 0	T(115)= 242 5	T(116)= 222 4
T(117)= 20 0	T(118)= 18 2	T(119)= 164 4	T(120)= 146 1
T(121)= 1 6	T(122)= 104 0	T(123)= 91 4	T(124)= 74 0

**Table A 7**  
(Sheet 4 of 4)

## TEMPERATURE DIFFERENTIALS

T( 1)= 740 8	T( 2)= 654 6	T( 3)= 574 4	T( 4)= 498 2
T( 5)= 424 2	T( 6)= 175 4	T( 7)= 147 3	T( 8)= 121 7
T( 9)= 97 9	T(10)= 75 2	T(11)= 741 7	T(12)= 655 9
T(13)= 576 3	T(14)= 501 3	T(15)= 429 3	T(16)= 417 3
T(17)= 177 4	T(18)= 170 6	T(19)= 144 6	T(20)= 120 2
T(21)= 97 1	T(22)= 75 0	T(23)= 744 2	T(24)= 659 3
T(25)= 561 2	T(26)= 506 9	T(27)= 441 1	T(28)= 394 5
T(29)= 167 7	T(30)= 166 5	T(31)= 187 0	T(32)= 162 5
T(33)= 119 3	T(34)= 117 0	T(35)= 95 5	T(36)= 74 7
T(37)= 747 4	T(38)= 663 7	T(39)= 587 7	T(40)= 518 2
T(41)= 454 9	T(42)= 397 2	T(43)= 283 3	T(44)= 350 9
T(45)= 208 4	T(46)= 196 0	T(47)= 199 8	T(48)= 175 7
T(49)= 154 8	T(50)= 134 1	T(51)= 113 7	T(52)= 93 0
T(53)= 74 4	T(54)= 750 5	T(55)= 667 8	T(56)= 593 6
T(57)= 526 6	T(58)= 467 0	T(59)= 413 9	T(60)= 368 2
T(61)= 347 7	T(62)= 220 7	T(63)= 320 4	T(64)= 300 8
T(65)= 296 2	T(66)= 277 0	T(67)= 274 9	T(68)= 257 6
T(69)= 256 4	T(70)= 240 4	T(71)= 240 1	T(72)= 223 7
T(73)= 225 6	T(74)= 206 4	T(75)= 212 9	T(76)= 188 1
T(77)= 169 0	T(78)= 149 7	T(79)= 130 3	T(80)= 111 2
T(81)= 92 4	T(82)= 74 2	T(83)= 752 6	T(84)= 670 0
T(85)= 597 7	T(86)= 532 5	T(87)= 474 6	T(88)= 423 8
T(89)= 379 0	T(90)= 342 7	T(91)= 311 5	T(92)= 285 1
T(93)= 262 4	T(94)= 241 9	T(95)= 222 6	T(96)= 203 7
T(97)= 184 6	T(98)= 165 9	T(99)= 147 0	T(100)= 128 2
T(101)= 109 7	T(102)= 91 6	T(103)= 74 0	T(104)= 753 4
T(105)= 671 8	T(106)= 599 1	T(107)= 534 5	T(108)= 477 2
T(109)= 427 0	T(110)= 383 5	T(111)= 346 3	T(112)= 314 7
T(113)= 287 7	T(114)= 264 0	T(115)= 242 5	T(116)= 222 4
T(117)= 203 0	T(118)= 163 9	T(119)= 164 9	T(120)= 146 1
T(121)= 177 6	T(122)= 109 3	T(123)= 91 4	T(124)= 74 0

Table A-8

$K = 52 \quad h_n = 812 \quad h_c = 1002 \quad h_o = 20000 \quad T_a = 70$   
G2 8 Loops

## TEMPERATURE DISTRIBUTION

TC (1) = 914		TC (2) = 700		TC (3) = 643		TC (4) = 451.5	
TC (5) = 210	TC (6) = 240	TC (7) = 152	TC (8) = 148				
TC (9) = 103	TC (10) = 74	TC (11) = 914	TC (12) = 764				
TC (13) = 715	TC (14) = 467	TC (17) = 304	TC (16) = 282				
TC (17) = 240	TC (18) = 229	TC (19) = 155	TC (20) = 144				
TC (21) = 104	TC (22) = 70	TC (23) = 916	TC (24) = 773				
TC (25) = 624	TC (26) = 493	TC (27) = 267	TC (29) = 275				
TC (29) = 242	TC (30) = 240	TC (31) = 242	TC (32) = 205				
TC (35) = 169	TC (34) = 125	TC (35) = 102	TC (36) = 70				
TC (37) = 913	TC (33) = 794	TC (39) = 656	TC (40) = 537				
TC (41) = 427	TC (42) = 312	TC (43) = 272	TC (44) = 265				
TC (45) = 245	TC (45) = 232	TC (47) = 244	TC (48) = 209				
TC (49) = 114	TC (50) = 154	TC (51) = 126	TC (52) = 97				
TC (53) = 70	TC (54) = 920	TC (55) = 793	TC (56) = 675				
TC (57) = 527	TC (57) = 472	TC (59) = 291	TC (60) = 330				
TC (61) = 264	TC (62) = 290	TC (63) = 260	TC (64) = 267				
TC (65) = 256	TC (66) = 256	TC (67) = 253	TC (68) = 251				
TC (69) = 250	TC (70) = 246	TC (71) = 240	TC (72) = 235				
TC (73) = 247	TC (74) = 226	TC (75) = 246	TC (76) = 209				
TC (77) = 199	TC (78) = 167	TC (79) = 143	TC (80) = 118				
TC (81) = 94	TC (82) = 70	TC (83) = 921	TC (84) = 800				
TC (85) = 630	TC (86) = 56	TC (87) = 498	TC (88) = 424				
TC (89) = 366	TC (90) = 323	TC (91) = 292	TC (92) = 273				
TC (93) = 257	TC (94) = 244	TC (95) = 230	TC (96) = 219				
TC (97) = 193	TC (97) = 179	TC (99) = 158	TC (100) = 150				
TC (101) = 114	TC (102) = 92	TC (103) = 70	TC (104) = 922				
TC (105) = 502	TC (106) = 692	TC (107) = 593	TC (108) = 507				
TC (109) = 434	TC (110) = 377	TC (111) = 313	TC (112) = 301				
TC (113) = 273	TC (114) = 260	TC (115) = 244	TC (116) = 229				
TC (117) = 212	TC (118) = 195	TC (119) = 176	TC (120) = 156				
TC (121) = 174	TC (122) = 113	TC (123) = 91	TC (124) = 70				

Table A 9

$$K = 52 \quad h_h = 925 \quad h_c = 1142 \quad h_o = 20000 \quad T_o = 70$$

G3 12 Loops

## TEMPERATURE DISTRIBUTION

T( 1)= 911 9	T( 2)= 79 2	T( 3)= 64 4	T( 4)= 452 3
T( 5)= 244 1	T( 6)= 240 2	T( 7)= 141 9	T( 8)= 148 6
T( 9)= 10 4	T( 10)= 70 0	T( 11)= 31 6	T( 12)= 762 7
T( 13)= 61 5	T( 14)= 40 0	T( 15)= 311 0	T( 16)= 284 0
T( 17)= 240 2	T( 18)= 223 6	T( 19)= 134 9	T( 20)= 144 6
T( 21)= 100 5	T( 22)= 70 4	T( 23)= 14 4	T( 24)= 771 7
T( 25)= 634 0	T( 26)= 449 4	T( 27)= 363 0	T( 28)= 276 3
T( 29)= 242 1	T( 30)= 240 4	T( 31)= 241 9	T( 32)= 204 7
T( 33)= 163 6	T( 34)= 135 4	T( 35)= 102 7	T( 36)= 70 0
T( 37)= 916 6	T( 38)= 732 7	T( 39)= 156 0	T( 40)= 537 2
T( 41)= 4 1	T( 42)= 232 9	T( 43)= 273 9	T( 44)= 266 3
T( 45)= 245 4	T( 46)= 23 2	T( 47)= 243 3	T( 48)= 200 3
T( 49)= 102 1	T( 50)= 154 7	T( 51)= 125 9	T( 52)= 97 7
T( 53)= 70 0	T( 54)= 318 5	T( 55)= 732 5	T( 56)= 674 9
T( 57)= 567 5	T( 58)= 47 3	T( 59)= 392 3	T( 60)= 331 0
T( 61)= 265 0	T( 62)= 290 5	T( 63)= 261 0	T( 64)= 267 7
T( 65)= 256 7	T( 66)= 257 0	T( 67)= 253 2	T( 68)= 251 7
T( 69)= 250 6	T( 70)= 246 5	T( 71)= 244 6	T( 72)= 233 4
T( 73)= 247 1	T( 74)= 226 0	T( 75)= 245 0	T( 76)= 202 3
T( 77)= 189 2	T( 78)= 166 8	T( 79)= 143 1	T( 80)= 118 7
T( 81)= 94 2	T( 82)= 70 0	T( 83)= 519 2	T( 84)= 799 0
T( 85)= 687 3	T( 86)= 586 4	T( 87)= 498 2	T( 88)= 424 7
T( 89)= 366 6	T( 90)= 327 0	T( 91)= 293 2	T( 92)= 273 1
T( 93)= 257 8	T( 94)= 244 4	T( 95)= 230 8	T( 96)= 215 6
T( 97)= 192 3	T( 98)= 179 2	T( 99)= 158 5	T(100)= 136 0
T(101)= 114 5	T(102)= 92 2	T(103)= 70 0	T(104)= 920 3
T(105)= 801 3	T(106)= 691 5	T(107)= 592 3	T(108)= 506 0
T(109)= 474 9	T(110)= 377 6	T(111)= 313 9	T(112)= 301 9
T(113)= 274 4	T(114)= 260 1	T(115)= 244 3	T(116)= 229 0
T(117)= 21 2	T(118)= 145 2	T(119)= 176 2	T(120)= 155 9
T(121)= 1 4 3	T(122)= 117 2	T(123)= 91 5	T(124)= 70 0

Table A 10

$$K = 52 \quad h_h = 1333 \quad h_c = 1846 \quad h_o = 140 \quad T_o = 25$$

Gf 6 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 925 2	T( 2) = 788 5	T( 3) = 644 3	T( 4) = 447 4
T( 5) = 272 2	T( 6) = 244 0	T( 7) = 126 4	T( 8) = 154 1
T( 9) = 115 3	T( 10) = 78 2	T( 11) = 32 7	T( 12) = 770 1
T( 13) = 517 2	T( 14) = 462 5	T( 15) = 244 3	T( 16) = 272 2
T( 17) = 242 9	T( 18) = 222 3	T( 19) = 125 9	T( 20) = 149 1
T( 21) = 112 1	T( 22) = 77 1	T( 23) = 326 3	T( 24) = 779 2
T( 25) = 528 7	T( 26) = 437 9	T( 27) = 222 2	T( 28) = 267 4
T( 29) = 24 0	T( 30) = 24 0	T( 31) = 244 8	T( 32) = 207 0
T( 33) = 171 7	T( 34) = 137 5	T( 35) = 104 5	T( 36) = 72 5
T( 37) = 92 5	T( 38) = 730 4	T( 39) = 559 7	T( 40) = 537 3
T( 41) = 425 1	T( 42) = 327 7	T( 43) = 255 4	T( 44) = 260 5
T( 45) = 247 0	T( 46) = 224 1	T( 47) = 146 0	T( 48) = 209 6
T( 49) = 132 4	T( 50) = 123 2	T( 51) = 124 8	T( 52) = 96 0
T( 53) = 67 4	T( 54) = 329 2	T( 55) = 300 3	T( 56) = 679 4
T( 57) = 562 1	T( 58) = 471 6	T( 59) = 320 1	T( 60) = 320 0
T( 61) = 280 1	T( 62) = 287 7	T( 63) = 257 0	T( 64) = 265 6
T( 65) = 254 2	T( 66) = 257 7	T( 67) = 252 0	T( 68) = 251 2
T( 69) = 250 2	T( 70) = 247 5	T( 71) = 249 1	T( 72) = 238 5
T( 73) = 240 1	T( 74) = 227 4	T( 75) = 247 2	T( 76) = 202 9
T( 77) = 190 1	T( 78) = 165 0	T( 79) = 140 4	T( 80) = 114 9
T( 81) = 99 1	T( 82) = 72 3	T( 83) = 334 7	T( 84) = 806 0
T( 85) = 692 2	T( 86) = 76 2	T( 87) = 496 3	T( 88) = 423 8
T( 89) = 265 0	T( 90) = 222 0	T( 91) = 292 2	T( 92) = 271 9
T( 93) = 256 8	T( 94) = 242 7	T( 95) = 234 0	T( 96) = 214 6
T( 97) = 197 5	T( 98) = 176 3	T( 99) = 155 3	T(100) = 132 5
T(101) = 102 2	T(102) = 7 2	T(103) = 60 8	T(104) = 931 0
T(105) = 303 1	T(106) = 626 6	T(107) = 325 5	T(108) = 507 7
T(109) = 42 7	T(110) = 28 3	T(111) = 22 7	T(112) = 300 4
T(113) = 271 1	T(114) = 252 1	T(115) = 243 2	T(116) = 227 9
T(117) = 211 4	T(118) = 193 2	T(119) = 172 2	T(120) = 152 3
T(121) = 120 0	T(122) = 106 2	T(123) = 72 2	T(124) = 52 9



Table A 11

$$K = 52 \quad h_n = 1333 \quad h_c = 1848 \quad h_o = 1 \quad T_o = 25$$

G1 8 Loops

## TEMPERATURE DISTRIBUTION

T( 1) = 925 2		T( 2) = 765 5		T( 3) = 602 3		T( 4) = 447 5	
T( 5) = 272 2	T( 6) = 249 8	T( 7) = 242 5	T( 8) = 247 3				
T( 9) = 246 2	T( 10) = 245 1	T( 11) = 245 7	T( 12) = 770 1				
T( 13) = 617 2	T( 14) = 462 6	T( 15) = 300 8	T( 16) = 272 3				
T( 17) = 249 8	T( 18) = 243 5	T( 19) = 243 3	T( 20) = 247 2				
T( 21) = 246 1	T( 22) = 245 0	T( 23) = 327 0	T( 24) = 779 3				
T( 25) = 636 8	T( 26) = 495 0	T( 27) = 362 3	T( 28) = 267 4				
T( 29) = 249 9	T( 30) = 249 9	T( 31) = 249 9	T( 32) = 249 0				
T( 33) = 248 0	T( 34) = 247 0	T( 35) = 246 0	T( 36) = 244 9				
T( 37) = 323 5	T( 38) = 730 5	T( 39) = 659 9	T( 40) = 537 6				
T( 41) = 425 5	T( 42) = 320 0	T( 43) = 255 4	T( 44) = 260 5				
T( 45) = 250 1	T( 46) = 254 1	T( 47) = 250 0	T( 48) = 249 7				
T( 49) = 248 9	T( 50) = 247 0	T( 51) = 247 1	T( 52) = 246 0				
T( 53) = 245 0	T( 54) = 329 0	T( 55) = 600 5	T( 56) = 679 8				
T( 57) = 569 7	T( 58) = 472 4	T( 59) = 331 1	T( 60) = 329 2				
T( 61) = 250 2	T( 62) = 287 9	T( 63) = 257 2	T( 64) = 267 0				
T( 65) = 254 6	T( 66) = 257 6	T( 67) = 252 6	T( 68) = 254 5				
T( 69) = 251 4	T( 70) = 253 6	T( 71) = 250 7	T( 72) = 253 0				
T( 73) = 250 3	T( 74) = 252 4	T( 75) = 250 1	T( 76) = 251 5				
T( 77) = 250 5	T( 78) = 249 5	T( 79) = 248 5	T( 80) = 247 4				
T( 81) = 246 3	T( 82) = 245 2	T( 83) = 336 7	T( 84) = 887 1				
T( 85) = 632 7	T( 86) = 539 7	T( 87) = 500 0	T( 88) = 425 6				
T( 89) = 367 4	T( 90) = 25 5	T( 91) = 297 5	T( 92) = 279 9				
T( 93) = 289 2	T( 94) = 262 7	T( 95) = 257 6	T( 96) = 255 7				
T( 97) = 253 5	T( 98) = 251 7	T( 99) = 251 7	T( 100) = 249 0				
T( 101) = 217 8	T( 102) = 246 2	T( 103) = 24 7	T( 104) = 931 4				
T( 105) = 43 4	T( 106) = 637 2	T( 107) = 36 5	T( 108) = 509 1				
T( 110) = 1 5	T( 110) = 279 3	T( 111) = 336 9	T( 112) = 307 1				
T( 113) = 3 3	T( 113) = 274 4	T( 115) = 256 4	T( 116) = 260 8				
T( 117) = 257 1	T( 118) = 274 4	T( 119) = 252 2	T( 120) = 250 7				
T( 121) = 249 2	T( 122) = 247 2	T( 123) = 246 6	T( 124) = 245 1				

Table A 12

K = 52  $h_h = 1223$ ,  $h_c = 1841$   $h_o = 140$   $T_o = 25$ 

G4 B Loops

## TEMPERATURE DISTRIBUTION

T	1) = 9.1 8	T	6) = 756 2	T	3) = 541 5	T	4) = 4.0 4
T 5) = 271 1		T 8) = 244 3		T 7) = 207 9		T 8) = 159 2	
T 9) = 116 7		T 10) = 61 0		T 11) = 922 5		T 12) = 760 7	
T 12) = 601 5		T 14) = 172 8		T 15) = 269 3		T 16) = 244 4	
T 17) = 246 4		T 18) = 244 6		T 19) = 194 9		T 20) = 153 5	
T 21) = 115 1		T 22) = 78 9		T 23) = 904 1		T 24) = 771 5	
T 25) = 625 0		T 26) = 164 0		T 27) = 144 8		T 28) = 264 6	
T 29) = 245 7		T 30) = 245 1		T 31) = 245 5		T 32) = 210 5	
T 33) = 175 4		T 34) = 140 8		T 35) = 106 9		T 36) = 74 0	
T 37) = 425 8		T 38) = 760 2		T 39) = 649 5		T 40) = 526 5	
T 41) = 117 9		T 42) = 329 6		T 43) = 262 6		T 44) = 271 1	
T 45) = 259 0		T 46) = 253 7		T 47) = 246 5		T 48) = 247 0	
T 49) = 247 3		T 50) = 232 8		T 51) = 246 6		T 52) = 211 7	
T 53) = 185 9		T 54) = 157 7		T 55) = 128 4		T 56) = 98 7	
T 57) = 69 6		T 58) = 427 0		T 59) = 791 7		T 60) = 666 5	
T 61) = 554 0		T 62) = 457 4		T 63) = 180 0		T 64) = 323 9	
T 65) = 288 1		T 66) = 251 2		T 67) = 267 4		T 68) = 251 9	
T 69) = 254 7		T 70) = 250 3		T 71) = 244 0		T 72) = 249 2	
T 73) = 211 5		T 74) = 115 2		T 75) = 185 2		T 76) = 172 1	
T 77) = 146 9		T 78) = 120 4		T 79) = 91 2		T 80) = 65 7	
T 81) = 97 4		T 82) = 794 6		T 83) = 172 4		T 84) = 563 3	
T 85) = 170 2		T 86) = 195 1		T 87) = 119 4		T 88) = 300 9	
T 89) = 215 4		T 90) = 157 7		T 91) = 112 4		T 92) = 227 7	
T 93) = 130 1		T 94) = 100 1		T 95) = 117 5		T 96) = 143 4	
T 97) = 117 1		T 98) = 94 1		T 99) = 114 6		T 100) = 114 6	



```

IF(R21 GE 0 01)N2POS=N2POS+1
NFOS=N1POS+N2POS
N1DFPO=N1FOS-N1
IF(R1 LT 0 01)N1DFPO=N1DFPO-1
N2DFPO=N2POS-N2
IF(R2 LT 0 01)N2DFFO=N2DFFO-1
GO TO 6
7 NFOS=NDV1
6 CONTINUE
IF(I LE 1)GO TO 5
N1DIF=N1-N1PAS
IF(R1PAS LT 0 01)N1DIF=N1DIF-1
N2DIF=N2-N2PAS
IF(R2PAS LT 0 01)N2DIF=N2DIF-1
5 CONTINUE
IFC=N1DIF+N2DIF
CALL D010
CALL D020
GO TO 60
50 ICH=ICH+1
NDIF=0
N=NDV1
IF(ICH NE 1)GO TO 55
NDIF=NDV1-(N1PAS-1)-(N2PAS-1)
55 CALL D030
IFC=NDIF
60 NANT=NANT+N+IPC
IFC=0
NPAS=N
N1PAS=N1
N2PAS=N2
L2PAS=L2
L3PAS=L3
R1PAS=R1
R2PAS=R2
100 CONTINUE
KK=0
JMAX=J-1
DO 70 I=1, JMAX
AUX=0
IF(IA(I, 2) LT 17)GO TO 80
KK=KK+1
AUX=D(KK)
80 WRITE(3, 11)(IA(I, K), K=1, 7), (B(I, L), L=1, 5), AUX
11 FORMAT(7I4, 6F12 5)
70 CONTINUE
STOP
END

```

```

SUBROUTINE DO18
COMMON ELOC01 ACEX RCIN RINT DCC TETA NDH NDV
COMMON ELOC02 IRCT06 TO E 700 50 D(120) N N1, N2, NANT MIDIF
      NCDIF ALFA BETA (N IT NDH1 I L2, C) J LT ID
COMMON ELOC03 ILIG RI RTHS L2M IJF
L1=NANT+1
L2=NANT+N1+MIDIF
L2M1 L2-1
DO 10 J=L1 L2
IF ILIG EQ 0 GO TO 14
ITIFO=15
ILIG=0
GO TO 17
14 ITIFO=1
IF I EQ 1 ITIFO=4
IF I EQ NCM1 ITIFO=5
IF J EQ L1 ITIFO=2
IF J EQ L1 AND I EQ 1 ITIFO=6
IF J EQ L1 AND I EQ NDH1 ITIFO=7
IF MIDIF GT 0 GO TO 16
IF RI LT 0 01 GO TO 15
IF J EQ L2M1 ITIFO=10
IF I EQ L2M1 AND I EQ 1 ITIFO=10
GO TO 15
16 DO 16 I=L1, N1DIF
L2M1=L2-I+2
IF I EQ L2M1 ITIFO=1
IF J EQ L2M1 AND I EQ 1 ITIFO=15
IF I EQ L2M1 ILIG 1
16 CONTINUE
15 IF J EQ L2 ITIFO=17
17 IA J 1)=J
IA J 2)=ITIFO
IA J 3)=J
IF ITIFO EQ 1 GO TO 10
IF ITIFO EQ 15 GO TO 17
E J 5)=RCIN+FLOAT T-1)*L1
IA J 4)=J+1
E J 1)=CS
IF J EQ L2-1 AND RI GE 0 01 E T 1)-RI
IF I LE L1 GO TO 18
IA J 6)=J-1
E J 2)=15
102 IF I LE 1 GO TO 10
IF J EQ L2-1 AND RIFAS 1) 0 01 GO TO 1050
DO 102 I=AFS-1 900
IF IA I)HRT T ME 1 06 16 I)HRT 11 17 GO TO 102
IA J 7)=I*HRT
GO TO 102
105 CONTINUE
1050 IA J 7)=1 10

```



```

IA(J, 7) = IA(IVIZ1, 7)
E(J, 2) = ALFA - EU
E(J, 4) = DT - E(J, 2)
GO TO 10
259 IA(J, 4) = IV1L
IA(J, 6) = IVIZ1
B(J, 3) = RLDALR(AU) * CCC * ALFA - CU
E(J, 1) = DE - E(J, 2)
GO TO 10
258 IA(J, 5) = IVIZ1
IA(J, 7) = IA(IVIZ1, 7)
E(J, 4) = DT
10 CONTINUE
RETURN
END

```







```

IF(JIC .GT. N DEB) B1 = 4
RU = MG DC( FINI( I) )
VFH = RLDALA( ALFA DC( AUX) PINT
CUX = (UX+04
EU = ANGALL(2 AUX DC( CUX)
VFV = AL( ALA( PUX DC( RU) ) FINI
IU = IO(1
I( ID) = AMINI( VEV VFH)
IVIZ = J 1
IF( VEV - VFH) 45 458 459
457 IAC( 5) = IVIZ + 2
DO 456 IVARRE = 1 808
IF( IAC( VARRE 5) NE IVIZ + 2 OR IAC( VARRE 2) GE 17) GO TO 454
IAC( 7) = IVARRE
GO TO 455
456 CONTINUE
455 B( J 2) = ALFA - PUX
B( J 4) = DT - B( J 2)
GO TO 28
459 IAC( J, 4) = IVIZ + 2
IAC( J, 6) = IVIZ
B( J, 1) = CUX - RLDALA( AUX DC( ALFA)
B( J, 3) = DS - B( J 1)
GO TO 28
458 IAC( J, 5) = IVIZ + 2
IAC( J 7) = IAC( IVIZ + 2 7)
B( J 4) = DT
28 CONTINUE
RETURN
END

```



```

IF I GT L1A J 6)E1
IR J = J+1
CU = FLOAT( (L5 - J + 1) * DS + RCIN
E J = -CU
516 F J 1) = H * F * ANG( C C ) AUX RINT
CONTINUE
IF J EQ L+2) IR J 6) = J+2
IF I GT L AND T LE L+2) I 5) = RCIN + FLOAT( L5 - NDJ ) * DS
GO TO 20
500 CONTINUE
LIJ = LIJ + 1
CU = FLOAT( J - 1) - L5 - (PLM - 1) * DS + RCIN
AU = ANG( C C ) RINT( CU )
EU = +LE * H * AUX( C C ) ALFA
ID = ID + 1
I J = +LE * H * ALFA( C C ) AUX( C C ) RINT
IVIZ = J - 1
IF EU - CU > 570 S#0 590
570 IR J 4) = IVIZ
IR J 6) = IR( IVIZ 6)
E J 1) = CU - EU
B J 2) = DS - E J 1)
IF( E J 2) GE 0 ) GO TO 5740
IR J 4) = 0
IR J 6) = 0
E J 1) = 0
E J 3) = 0
IVIZ = J - 1
IVIZ1 = IR( IVIZ 6)
IR J 5) = IVIZ1
IR J 7) = IR( IVIZ1 7)
DUX = CU - DS
EUY = ANGALL( 1 AU ) C C ) CU )
I( ID ) = +LE * H * ALFA( C C ) AUX( C C ) RINT
E( J 2) = ALFA - EUY
AU = ANG( C C ) RINT( DUX )
EUY = ANGALL( 1 AU ) C C ) DUX )
E( J 4) = EUY - EUY
IF( E( J 2) + B( J 4) GE DT * 0.01 * E( J 4) ) DT F J 2)
5 00 GO TO 30
500 IR( J 1) = IR( IVIZ 1)
IR J 6) = IVIZ
E J 3) = EU - CU
B J 1) = DS - E J 3)
IF( E J 1) GE 0 ) GO TO 5450
IR J 4) = 0
IR J 6) = 0
B J 1) = 0
B J 2) = 0
IVIZ = J - 1
IR J 5) = IVIZ + 2
DO 5456 I VAPPE = 1 700
IF( IR( I VAPPE 5) NE IVIZ + 2 ) IR I VAPPE 2) GE 1 ) GO TO 5456
IR J 7) = I VAPPE
GO TO 545E
545E CONTINUE
IR J 7) = IR J 5) + 1
DUX = CU + DS
EUY = ANGALL( 2 AU ) C C ) CU )
I( ID ) = +LE * H * ALFA( C C ) AUX( C C ) RINT
B J 2) = ALFA - EUY
AU = ANG( C C ) RINT( DUX )
EUY = ANGALL( 2 AU ) C C ) DUX )
E J 4) = EUY - EUY
GO TO 10
545F I U - CU + DS

```

```
      1)  HROHLI < AU< OCC, DU> >  
      2)  H< FLIHLA< EU>, OCC, AU> >-RINT  
      3)  J, 1< ALFA< EU>  
      4)  J, 4< DT< E< J, 2>  
004  GO TO 20  
000  IA< J, 4>=IVIZ  
      IA< J, 6>=IA< IVIZ, 6>  
      F< J, 2>=DE  
_0  CONTINUE  
      RETURN  
      END
```

```

SUBROUTINE ENTRA
  LOGICAL*1 (C 72)
  COMMON FLOOD1 RYEN RCON RINT DCC TETA NDH NDV NS DT ALFA
  CALL ERASE
  WRITE(6 1)
1  FORMAT (1X 'ENTRE OS RAIOS DAS CIRCUMF. CONCENTRICAS '
  *      1X 'E TERMA INTERNA E DA NAO CONCENT
  *      REC RCON RINT) > /)
  CALL BELL
  READ(6 2)CC
2  FORMAT(22A1)
  P=0
  CALL REALFF(F CC 72 RCI)>
  CALL REALFF(F CC 72 RCON)
  CALL REALFF(F CC 72 RINT)
  WRITE(6 3)
3  FORMAT(1X 'ENTRE A DISTANCIA DO CENTRO DAS CIRC. CONC. AO '
  *      1X 'CENTRO DA CIRC. NAO CONC. E O ANGULO EM GRAUS
  *      (DCC TETA) > /)
  CALL BELL
  READ(6 2)CC
  P=0
  CALL REALFF(F CC 72 DCC)
  CALL REALFF(F CC 72 TETA)
  WRITE(6 4)
4  FORMAT (1X 'ENTRE O NUMERO DE DIVISOES NA HORIZONTAL E '
  *      1X 'O NUMERO DE DIVISOES NA VERTICAL (NDH NDV) > ,//)
  CALL BELL
  READ(6 2)CC
  P=0
  CALL INTFF(F CC 72 NDH)
  CALL INTFF(F CC 72 NDV)
  RETURN
END

```

### C CHALST FTH

C

```

  DIMENSION IA(7) B(6)
  DATA DISTMP CRDISTMP /
  WRITE(5 3)
3  FORMAT(20X 'GRID CONFIGURATION' )
  WRITE(5 4)
4  FORMAT(7X C1 2X C2 2X C3 2X C4 2X C5 2X
  *      C6 3X C7 5X C8 5X C9 5X C10 4X C11 4X C12 //)
  N=1
  GO TO 100
101 N=1
  WRITE(5 5)
5  FORMAT(1 // / 7X C1 2X C2 2X C3 2X C4 2X C5'
  *      2X C6 3X C7 ' ) C6 5X C9 5X C10 4X C11' 4X C12 //)
100 READ(3 1 ENL=200) IA 1) I=1 7) (B) J) J=1 6)
1  FORMAT 7(4 6) I. '

```

```
      E(2)=180 *B(2)/3 141E
      E(4)=180 *F(4)/2 141E
      WRITE(5,2)((IA(I), I=1,2), (IA(I), I=4,7), (E(J), J=1,6)
      FORMAT(5X,6I4,6F7.2)
      IF(N EQ 45)GO TO 101
      N=N+1
      GO TO 100
200  CALL RESUME(DISTMF)
      CALL E IC
      END
```

```

FUNCTION CORIA ALFA DIST)
COMMON EIDOU1 FCEX RFIN RINT OCE TETA NCH NOV
EQ=EQ*(1/PI)*ALFA)
IF(EO GE RINT)GO TO 10
CORIA = SQRT(RINT**2-EO**2)
DIST=DEC*(COS ALFA)-CORIA /
GO TO 100
10 CORIA O
LIST=RCEX
100 RETURN
END

```

```

FUNCTION ANG X Y Z
S=1/2*(Y+Z)
R=SQRT( (X**2)-(S**2) )
F=F (S-T)
ANG=ATAN R
RETURN
END

```

```

FUNCTION ANGALL(ISOI ANGO1 A1 A
CF=(A2*(1 + SIN ANGO1/2) + COS ANGO1/2)**2)**1/2
A1+(SIN ANGO1/2)/COS(ANGULO/2))
AUX1= (1+SQRT(CF**2-4))**2
AU = (1-SQRT(CF**2-4))**2
IF(ISOI EQ 1)AU =AMAX(AUX1 AU/2)
IF(ISOI EQ 2)AUX= (MIN(AUX1 AU/2)
14001 1410-ANL 142 *ATAN AU
1410)
END

```

```

17 1 FLENE P1 (H1) ANG2)
18 F (O) P (IN (1410-PI 1-ANG2)
19)
20)

```



```

SUBROUTINE FSCALE (IF, C, LIM, F
LOGICAL*1 C, LIM
F=0
IA=0 1
I IF
IS=0
10 I=I+1
IF I GE LIM GO TO 20
IF C(I) EQ 0 GO TO 10 20
IF C(I) LT 'E0 OF I' GO TO 10 20
F=10 *F+C(I) I="E0"
JS=IS+1
I=I+1
IF I GE LIM GO TO 20
IF C(I) EQ "SE" GO TO 20
IF C(I) GE 'OPANE (I)' (F '1,2' GO TO 20
GO TO 20
20 I=I+1
JS=JS+1
IF I GE LIM GO TO 20
IF C(I) LT '0 OF I' GO TO 20 20
F=F+F0+C(I) I="0"
FA=IH+0 1
GO TO 20
30 I=F-IS*L
IF JC LE 000 0 10
IF C(JC) EQ 'E' F=1
40 IF=I
50 RETURN
END

```

```

                JIJIF TTAGT
                NIJII QIUGS ACUNIS UJSEIS

MOVF      MOV R0, R1
MOV      MOV R2, R1
RDE     R0, R1
CLR     CLR R1
SHIF     (R0, R1) #R5
        FGT DONE
        INCL R1
        CMPL R1, #0
        BJT NOTOCT
        CMPL R1, #0
        BJT OCT
NOTOCT    CMPL R0, R1
        BJT SHIF
NEGT     (R0, R1) #R1
        CMPL R0, R1
        BGE DONE
        MOV R1, R0
        CMPL R1, #0
        BJT COME
        CMPL R1, #0
        FGT DONE
OCT      RSL R0
        RSL R1
        RSL R2
        FIC #1000, 0, 1
        BJS FT, R2
        BR NEGT
COME     MOV R0, R1
        MOV R1, (R0)
        RTS IC

INTEF    MOV @R0, R0
MOV     MOV (R0), R1
CLR     CLR R3
CLR     CLR R4
RDE     R0, R1
CLPE    NEGA
FI      CMPL R0, R1
        BGE F2
        CMPL (R1), #0
        BEO MEND
        CMPL R1, #0
        BJT F1
        CMPL R1, #0
        BJT NUMER0
F1      CMPL (R0), R1
        BR FT
F2      INCL NEGA
        (R0, R1) #R1
        CMPL R0, R1
        BGE F1
        CMPL R1, #0
        BJT F1
        (R1, R1) #1

```

```

NUMERO      EGT E 11
            MOV F3, F
            LMF R2, M
            FGE NXT
            CLC
            FOL F2
            FOL R2
            ADD F4, F2
            FOL F2
            MOVB (R1), FC
            FIC #177700, FL
            ADD FC, FL
            EMI NXT
            MOV F2, F4
            EF NXT
{ } II
            TSTE NEGA
            BEQ EC
            NEG F4
L2          MOV F0, 0 (F5)
            MOV F4, 010 (F5)
            RTS FC
NEGA        BYTE 0
            EVEN
,
EALL        ALUN$E #6, #"TT, #0
            QIO$E #10 ATT, #6
            QIO$E #10 WLE, #6, #2, , , #MSG #1
            WTSE$E #2
            QIO$E #10 DET, #6
            RTS FC
MSG         ASCII <7
            EVEN
FILE       ALUN$E #C #"TT, #0
            QIO$E #10 ATT, #6
            QIO$E #10 WLE, #6, #2, , , <#MSG1, #6, >
            WTSE$E #2
            QIO$E #10 DET, #6
            RTS FC
FILE1      ASCII <35 <37><37><27><37><37>
            END

```

```

C      MONTH FTH
C      MONTE THE MATRIX OF COEFFICIENTS USING THE OUTPUT OF CHACHA
C
      DIMENSION IA(7) E(6) ( 5) 10 5
      COMMON RL(700)
      DATA DISTNF,6ADISTNF,
      REAC ( 2 2)BL
232    FORMAT(4F12 5)
      DO 20 I=1 700
      RL I)=RLAM(I)
20    CONTINUE
      READ(4 234)HC HH HO TC TH TD
234    FORMAT(6F12 5)
900    REAL( 81 END=1000)(IA(I) I=1 7) (E(J) J=1 6)
81    FORMAT(7I4 6F12 5)
      DO 50 I=1 5
      C(I)=0
      IC(I)=0
50    VB=0
      GO TO(1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18)IA(I)
1    IC(1)=IA(7)
      IC(2)=IA(6)
      IC(3)=IA(5)
      IC(4)=IA(4)
      IC(5)=IA(3)
      DS=B(1)
      DT=B(2)
      C(1)=(RL(IA(2))+RL(IA(7)))/(2 *E(5)+DT**2)
      C(2)=(B(5)*2 -DS)*(RL(IA(6))+RL(IA(3)))/(4 *DS**2)
      AUX=((B(5)+2 *DS)+(RL(IA(3))+RL(IA(4)))+(
      * (B(5)+2 -DS)*(RL(IA(6))+RL(IA(3)))))/(4 *DS**2)
      BUX=(RL(IA(5))+RL(IA(3))+2 *RL(IA(7)))/(2 *B(5)+DT**2)
      C(3)=-1 *(AUX+EUX)
      C(4)=(B(5)+2 *DS)*(RL(IA(3))+RL(IA(4)))/(4 *DS**2)
      C(5)=(RL(IA(5))+RL(IA(3)))/(2 *E(5)+DT**2)
      VB=0
      GO TO 800
2    CONTINUE
6    CONTINUE
7    IC(1)=IA(3)
      IC(2)=IA(4)
      DS=B(1)
      C(1)=-HH RL IA 3) DS
      C(2)=RL(IA(3)) DS
      VE=-HH*TH
      GO TO 600
3    CONTINUE
8    CONTINUE
9    IC(1)=10 5)
      IC(2)=IA(3)

```

```

      C(1)=RL(IA(3)) DS
      C(2)=-HD RL(IA(3))/DS
      VE=H0*H1
      GO TO 300
4     IC(1)=IA(6)
      IC(2)=IA(7)
      IC(3)=IA(4)
      IC(4)=IA(5)
      C(1)=E(1)
      C(2)=B(2)
      C(1)=(1/DS**2)*(B(5)-DS/2)*( (RL(IA(6))+RL(IA(3)))/2 )
      C(2)=(1/DS**2)*(B(5)+DS/2)*( (RL(IA(4))+RL(IA(3)))/2 )
      C(4)=(RL(IA(3))+RL(IA(5)))/(E(5)*DT**2)
      AUX=(E(5)+DS/2)*( (RL(IA(4))+RL(IA(7)))/2 )+
          (B(5)-DS/2)*( (RL(IA(6))+RL(IA(3)))/2 )*(-1/DS**2)
      BU=C(4)
      C(2)=AUX-BUX
      VE=0
      GO TO 500
5     IC(1)=IA(7)
      IC(2)=IA(6)
      IC(3)=IA(2)
      IC(4)=IA(4)
      C(1)=E(1)
      C(2)=E(4)
      C(1)=(RL(IA(7))+RL(IA(3)))/(B(5)+DT**2)
      C(2)=(1/DS**2)*(B(5)-DS/2)*( (RL(IA(6))+RL(IA(3)))/2 )
      AUX=(B(5)+DS/2)*( (RL(IA(4))+RL(IA(2)))/2 )+
          (E(5)-DS/2)*( (RL(IA(6))+RL(IA(3)))/2 )*(-1/DS**2)
      BU=C(1)
      C(3)=AUX-BUX
      C(4)=(1/DS**2)*(C(1)+DS/2)*( (RL(IA(4))+RL(IA(3)))/2 )
      VE=0
      GO TO 800
10    CONTINUE
11    CONTINUE
12    CONTINUE
15    CONTINUE
16    IC(1)=IA(7)
      IC(2)=IA(6)
      IC(3)=IA(2)
      IC(4)=IA(4)
      IC(5)=IA(5)
      DS1=B(3)
      DS2=B(1)
      DT1=B(4)
      DT2=B(7)
      C(1)=(1/(DT1+B(5)+(DT1+DT2)))*( (RL(IA(3))+RL(IA(7)))/2 )
      C(2)=(1/((DS1+DS2)+DS1))*(B(5)-(DS1/2)+
          (RL(IA(3))+RL(IA(6)))/2 )
      C(4)=(1/((DS1+DS2)+DS2))*(B(5)+DS2/2)+
          (RL(IA(3))+RL(IA(4)))/2 )
      C(5)=(1/(DT2+B(5)+(DT1+DT2)))*( (RL(IA(3))+RL(IA(5)))/2 )
      C(3)=-C(4)-C(2)-C(5)-C(1)
      VB=0
      GO TO 800
17    IC(1)=IA(6)
      IC(2)=IA(3)
      IC(3)=IA(4)
      IC(4)=IA(5)
      DE=E(7)
      DT=F(2)
      LF=L(1)
      C(1)=(1/(C*(F)+DE))*( (RL(IA(6))+RL(IA(3)))/2 )*(B(5)-DS/2 )
      C(2)=(1/(C*(F)+DE))*( (RL(IA(4))+RL(IA(2)))/2 )*(B(5)+DS/2 )

```

```

173 C(1)=(RL(IA(3))+RL(IA(5)))/2 *(P(5)*DT**2)
174 AU=(RL(IA(3))+RL(IA(4)))/2 *(B(5)+P(2))/DP+
175 (RL(IA(3))+RL(IA(5)))*(E(5)-D(5))/DS)
176 IF(IA(4)
177 C(1)=-1/(P(5)+DP)*AU-D(5)
178 VE=0
179 GO TO 180
180 IC(1)=IA(3)
181 IC(2)=IA(4)
182 IC(3)=IA(5)
183 IC(4)=IA(6)
184 IC(5)=IA(7)
185 IF(IA(4)
186 IF(IA(5)
187 C(1)=-1/(P(5)+DP)*(RL(IA(3))+RL(IA(6)))/2 *(B(5)-D(
188 C(2)=-1/(P(5)+DP)*(RL(IA(3))+RL(IA(4)))/2 *(B(5)+D(
189 C(3)=-1/(P(5)+DP)*(RL(IA(5))+RL(IA(6)))/2 *(B(5)*DT**2)
190 AU=(RL(IA(3))+RL(IA(4)))/2 *(E(5)+D(5)/2)/DS+
191 (RL(IA(5))+RL(IA(6)))/2 *(P(5)-DP/2)/DP)
192 IF(IA(4)
193 C(1)=-1/(P(5)+DP)*AU-D(5)
194 VE=0
195 GO TO 180
196 CONTINUE
197 IF(IA(4) EQ 0 AND IA(5) EQ 0 GO TO 178
198 IC(1)=MIN0(IA(3), IA(4), IA(6))
199 IC(2)=MA0(IA(3), IA(4), IA(6))
200 IF(IA(1) EQ 0)
201 IF(IA(5) EQ 0) GO TO 70
202 IF(IA(1) EQ IC(1) OR IA(1) EQ IC(3)) GO TO 70
203 IC(2)=IA(1)
204 CONTINUE
205 DS1=P(2)
206 DS2=P(1)
207 AUX=(DS2*RL(IA(3)))/(B(6)*(1+(P(2))))
208 EU=(DS1*RL(IA(3)))/(E(6)*(DS1+DS2))
209 CUX=-1*(RL(IA(3)))/B(6)+HC)
210 IF(IA(1) EQ IA(3)) C(1)=CUX
211 IF(IA(1) EQ IA(6)) C(1)=AUX
212 IF(IA(2) EQ IA(3)) C(2)=CUX
213 IF(IA(2) EQ IA(4)) C(2)=EUX
214 IF(IA(2) EQ IA(6)) C(2)=AUX
215 IF(IA(3) EQ IA(3)) C(3)=CUX
216 IF(IA(3) EQ IA(4)) C(3)=EU
217 GO TO 179
218 IC(1)=MIN0(IA(3), IA(5), IA(7))
219 IC(2)=MAX0(IA(3), IA(5), IA(7))
220 DO 60 K=1, 2
221 IF(IA(K) EQ IC(1) OR IA(K) EQ IC(2)) GO TO 60
222 IC(2)=IA(K)
223 CONTINUE
224 DT1=B(4)
225 DT2=B(2)
226 AU=(DT2*RL(IA(3)))/(B(6)*(DT1+DT2))
227 EU=(DT1*RL(IA(3)))/(B(6)*(DT1+DT2))
228 CUX=-1*(RL(IA(3)))/B(6)+HC)
229 IF(IA(1) EQ IA(3)) C(1)=CUX
230 IF(IA(1) EQ IA(7)) C(1)=AUX
231 IF(IA(2) EQ IA(3)) C(2)=CUX
232 IF(IA(2) EQ IA(5)) C(2)=EUX
233 IF(IA(2) EQ IA(7)) C(2)=AUX
234 IF(IA(3) EQ IA(3)) C(3)=CUX
235 IF(IA(3) EQ IA(5)) C(3)=EUX
236 VE=-HC+TC
237 ICOUNT=0
238 DO 60 K=1, 5

```



```

C      FINA FTR
C      CALLS SPAMAT AND PRINTS THE SOLUTION VECTOR
C
COMMON H(140,5) IC(140,5) INL(140) B(140),X(140)
DATA D1,TMP, &ALI,TMP /
J=1
10  READ(1,1) END(=0) A(J,J) J=1(=0) P(J) IC(I,J) K=1(=5) INZ(I)
1  FORMAT(4F12.5,11)
   I=I+1
   GO TO 10
20  N=I-1
   DO 30 I=1,N
   X(I)=I
30  CONTINUE
   CALL SPAMAT(N)
   WRITE(2,3)X
3  FORMAT(4F12.5)
   WRITE(5,4)
4  FORMAT(20X TEMPERATURE DISTRIBUTION )
   DO 40 I=1,N,4
   K=I+3
   IF(K GT N) EN
   WRITE(5,2)((L,X(L)) I=I,K)
2  FORMAT('  9) 4) 1) 1) )= F( 1 3X))
40  CONTINUE
   CALL RESUME D1,TMP)
   CALL EXIT
END

```



```

C      SUBROUTINE SFMATH
C      SOLVE THE LINEAR SYSTEM MATRIX TECHNIQUE A SYSTEM OF
C      LINEAR EQUATIONS BY GAUSS ELIMINATION METHOD
C
C      SUBROUTINE SFMATH (N)
COMMON N (140) Z (1) IC(140) INZ(140) B(140) X(140)
TOL = 0
N = N - 1
DO 10 I = 1, N - 1
  NZ = INZ(I)
  AM = A(I, I)
  I = I + 1
  DO 14 K = I + 1, N
    IF (ABS(A(I, K)) - ABS(AM)) > 10 * TOL
      AM = A(I, K)
      NZ = K
14  CONTINUE
    IF (IC(I, NZ) EQ 0) GO TO 41
    CALL RUCCOL(I, N, NZ)
    A1 = A(I, NZ)
    IF (ABS(A1) > TOL) GO TO 101
101  WRITE (*, 102) I, A1
102  FORMAT (10X, NA, LINHA, I7, 5X, VALOR MAX = E8.4)
24  CALL TRANS(K2, I)
    B(I) = B(I) / A1
    DO 20 J = I + 1, N
      A(I, J) = A(I, J) / A1
    DO 40 J1 = I + 1, N
      IF (INZ(J1) GT NZ) NL = INZ(J1)
      NZ = INZ(J1)
      DO 41 K = 1, NZ
        IF (IC(J1, K) EQ 0) GO TO 42
41  CONTINUE
        GO TO 40
42  A2 = A(I, K)
        CALL TRANS(K, I)
        NZ = INZ(I)
        NZ1 = INZ(I1)
        DO 50 K1 = 1, NZ1
          DO 60 J = 1, NZ
            IF (IC(I1, J) EQ 0) GO TO 61
60  CONTINUE
            GO TO 50
        A(I1, J) = A(I1, K1) - A2 * A(I, J)
70  CONTINUE
        NZ1 = INZ(I1)
        DO 70 K = 1, NZ
          DO 80 K1 = 1, NZ1
            IF (IC(I, J) EQ 0) GO TO 0
            IF (IC(I1, K1) EQ 0) GO TO 0
            CONTINUE
            DO 71 J1 = 1, NZ1
              IF (IC(I1, J1) EQ 0) GO TO 72
71  CONTINUE
              INZ(I1) = INZ(I1) + 1
              I1 = I1 + 1
              NZ1 = NZ1 + 1

```

```

72      A(I1,K1)=-A2*A(I,K)
       IC(I1,K1)=IC(I,K)
76      CONTINUE
       E(I1)=E(I1)+A1*(E(I))
40      CONTINUE
10      CONTINUE
       E(N)=E(N)+A*(N-1)
       DO 90 J=1,N-1
         IE=N-J
         IA=N
         NZ=INZ(IE)
         DO 90 M=1,J
           DO 91 K=1,NZ
             IF( IC(IE,K) EQ IA)GO TO 92
91          CONTINUE
           GO TO 93
92          E(IE)=E(IE)+A*(IE)*E(IA)
93          IA=IA-1
96          CONTINUE
         DO 94 I=1,N
           L=IFIX(X(I))
           X(I)=E(L)
94          CONTINUE
       RETURN
       END

```

```

SUBROUTINE INDCOL(K1 N K2)
COMMON A(140 25) IC(140 25) INZ(140) B(140) X(140)
K=1
IF(1-IC(1 1) .EQ. 0)
  N=INZ(1)
  DO 10 I=1 N2
    IF(1-IC(1 I)) EQ 1) GO TO 31
10 CONTINUE
  GO TO 11
21 IF(1-IC(1 1) .EQ. 0)
30 IC(1 I)=1
32 DO 11 I=1 N
  IF(X(I) .EQ. FLUAT(10)) GO TO 1
  GO TO 11
12 N=N+1
  GO TO 11
14 IF(N .EQ. 0) FLUAT(10)=X(1)-IC(1
16 CONTINUE
  DO 10 I=1 N
  IF(1-IC(1 I)) GO TO 10
  N2=INZ(I)
  DO 20 K=1 N2
  IF(1-IC(I K)) EQ 1) GO TO 21
  IF(1-IC(I K)) EQ 1) IC(I K)=1
  GO TO 20
  IC(I K)=IC(1
18 CONTINUE
19 CONTINUE
RETURN
END

```

```

SUBROUTINE TRANSCK(I)
COMMON A(140 25) IC(140 25) INZ(140) B(140) X(140)
N2=INZ(I)
K1=K+1
IF(K1 .GT. N2) GO TO
DO 10 K2=1 N2
  IC(I K2)=IC(I K1)
  A(I K2)=A(I K1)
  K1=K+1
10 CONTINUE
  IC(I K2)=0
  A(I K2)=0
  GO TO 10
20 IC(I K)=0
  A(I K)=0
  INZ(I)=INZ(I)-1
  IF(INZ(I) .EQ. 0) INZ(I)=1
RETURN
END

```





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